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Thermal Modeling of a Spiral Planar Inductor

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Thermal Modeling of a Spiral Planar Inductor

Melati Rabia^{1*}, Hamid Azzedine¹, Adda Benattia Tekkouk².

Abstract — In the literature, it is common practice to use well-known and particularly robust numerical methods, such as the finite difference method and the finite element method for calculating the temperature in electronic components or devices.

To calculate the operating temperature of a spiral planar inductor, we will adopt a method that is simpler to implement and gives equally reliable results. This method is based on the design of the thermal circuit of the inductor.

The design of the thermal circuit results from the decomposition of the inductor into volume elements V_i and the similarity between the thermal and electrical circuits.

By applying the nodal method, each volume defines a thermal sub-circuit. The assembly forms a complicated global circuit. This circuit is simplified into a " π " thermal circuit.

The " π " thermal circuit associated with the Fourier's law facilitates calculation of the temperatures in different areas of the inductor.

The obtained results are good and are validated by the simulation of heat flux. This simulation is performed with FEMLAB software.

Keywords — Spiral planar inductor – Thermal circuit – Integrated inductor – Thermal modeling– Heat transfer -Thermal effect.

1 INTRODUCTION

In microelectronics, the involvement of more and higher power dissipation on very restricted surfaces can cause considerable heating. Thus, the problems associated with the heat dissipation in the electronic systems are of great importance.

Thermal effects in power electronics systems can manifest in different ways, either by a drift of components temperature resulting in significant variations in the electrical performance, or by a rise in temperature which can cause welds connecting the components due to dimensional variations.

The nature of materials used in the manufacture of electronic components, in turn requires a working temperature limit, because a poorly managed operating temperature can lead to a change of state of materials, such as degaussing of magnetic materials, or the fusion of conductive materials. Therefore, to ensure the good operation and longevity of an electronic component, the operating temperature must be well controlled [1-6].

2 PRESENTATION OF THE SPIRAL PLANAR INDUCTOR

The micro-coil selected is a square planar spiral type, its geometrical parameters are respectively: the internal and external diameters d_{in} and d_{out} , the number of turns n , the inter-turn spacing s , the width w and the thickness t of the conductor (figure 1).

Figure1. Geometrical parameters of the square spiral planar inductor

The copper spiral is placed on a ferromagnetic core. The core is a parallelepiped block of thickness h_{mag} . This spiral is separated from the core by an insulating layer of silicon dioxide of thickness t_{ox} . The assembly is placed on a semiconductor substrate with h_{sub} thickness (figure2).

Figure 2. Cross section of the spiral planar inductor

This inductor is dimensioned to be integrated into a buck DC / DC micro- converter (Figure3).

Figure 3. Schematic diagram of the buck converter

We choose the schedule of conditions according to:

Input voltage: $V_{in} = 5V$

Output voltage: $V_{out} = 5V$

Average output current: $I_{out} = 0.38A$

Maximal current: $I_{Lmax} = 0.6A$

Operation frequency : $f = 1.5 MHz$.

The dimensioning of the inductor is performed with the dimensioning method presented by the authors in a previous works [7]. It should be noted that this dimensioning is performed so as to attenuate the skin effect considerably.

The planar inductor is traversed by a current of maximum value $I_{Lmax} = 0.6A$ and minimum value $I_{Lmin} = 0.16A$. The calculated value of the inductance is $L = 1.89\mu H$ [7].

The dimensioning results are summarized in Table I [7].

**TABLE I
VALUES OF THE INDUCTOR GEOMETRICAL PARAMETERS**

Geometrical parameters	Values (μm)
<i>Number of turns : n</i>	2
<i>Average conductor length : L_{avg}</i>	4402,5
<i>Conductor width : w</i>	120
<i>Conductor thickness : t</i>	20
<i>Inter-turn spacing: s</i>	97,5
<i>External diameter of the spiral: d_{out}</i>	900
<i>Internal diameter of the spiral: d_{in}</i>	225
<i>The thickness of the core: I_{Lmag}</i>	640
<i>The thickness of the insulating layer: t_{ox}</i>	60
<i>The thickness of the substrate: h_{sub}</i>	100

Our objective is to calculate the operating temperature of the planar inductor using a simple and reliable method. To do this, we will first design the thermal circuit of the inductor.

3 THERMAL MODEL OF SPIRAL PLANAR INDUCTOR

In the literature, generally, we use robust numerical methods, such as finite difference method and the finite element method to calculate the temperature in components or devices electronic [8].

In this paper, we will adopt a method that is simpler to implement and gives equally reliable results. This method is based on the design of the thermal circuit.

To design the thermal circuit of our spiral planar inductor, we propose an original approach, based on the nodal method [9] [10] [11]. This thermal circuit makes it possible to determine the operating temperature of our component.

3.1 The nodal method

The nodal method makes it possible to discretize the medium to be studied into elements of volumes V_i assumed with an isothermal temperature T_i . Each volume V_i is characterized by a thermal capacity C_i and a heat source P_i . The gravity centres of the volumes V_i called nodes are interconnected (Figure 4) [11] [12].

Figure 4. Three-dimensional mesh network

3.2 Thermal model of the spiral planar inductor

The main objective of the thermal modeling of the spiral planar inductor is to obtain a simple and reliable model allowing to determine its operating temperature. This temperature depends on Joule effect losses and iron losses. The Joule effect losses in the conductive spiral are the main source of heat.

3.2.1 Simplifying assumptions

The development of the thermal model of the inductor is based on the following simplifying assumptions:

- The dimensions of our inductor are very small (a few micro meters), and there is no fluid matter in the inductor, so the convective heat transfer is negligible.
- We assume that the heat transfer by conduction is unidirectional.
- The surfaces of the stacked layers are assumed to be very smooth, so the thermal capacities at the interfaces are negligible.

3.3 Design of the inductor thermal circuit

To design the thermal circuit using the nodal method, we discretize our inductor into volume elements V_k . We first discretize the spiral into nine segments of sections S_k ($k=1, \dots, 9$), and of volumes $V_{l,k}$ ($k=1, \dots, 9$). The gravity center of each volume $V_{l,k}$ is a node (Figure 5).

Figure 5. The different sections S_k and different volumes $V_{l,k}$ of the spiral

The cross section of the inductor shows the different thermal resistances R_{th} and thermal capacities C_{thi} ($i=1, \dots, 3$) and C_{vi} ($i=1, \dots, 5$) associated with each node (Figure 6).

Figure 6. Cross section of the inductor, different thermal resistances and capacities associated with a node

Note that the overall volume of the conductive ribbon of the inductor is the main source of heat, and volumes $V_{l,k}$ ($k=1, \dots, 9$) are sub-sources of heat. In each volume, the heat is transferred to the three layers of materials stacked below. Applying the nodal method, we consider that the node of each volume $V_{l,k}$ (Figure 7) is associated with a thermal sub-circuit (Figure 8).

Figure 7. Discretization of the conductive ribbon into volumes associated with their nodes

The cross section of the inductor (Figure 6), the discretization of the conductive ribbon into volumes $V_{l,k}$ associated with their nodes (Figure 7) and the similarity between the electrical and thermal phenomena, allowed us to design an overall thermal model of the inductor (Figure 8).

Figure 8. Overall thermal circuit of the inductor

The thermal sub-circuits in the first four nodes are then connected in series. It is the same for the sub-circuits of the last four nodes. The whole is modelled by a "π" thermal circuit (Figure 9).

Thermal capacities between the segments must also be taken into account.

Note that the model considers just the thermal capacities between the adjacent parallel segments. This approximation is valid because the thermal capacities between the other segments are negligible.

Figure 9. The "π" Thermal circuit of the spiral planar inductor

C_{ths} is the equivalent thermal capacity of inter-turns thermal capacities.

$R_{jthc}, R_{jthox}, R_{jthp}, R_{jthsi}$ ($j=1, \dots, 2$) are the respective equivalent thermal resistances of the four first and four last nodes in the respective layers: conductor (copper), insulating (silica), core (permalloy) and substrate (silicon).

$C_{1-2}, C_{2-3}, C_{3-4}$ and $C'_{1-2}, C'_{2-3}, C'_{3-4}$ are the respective equivalent thermal capacities of the four first and four last nodes in the three respective interfaces: copper-insulating, insulating-core and core-substrate.

4 TEMPERATURE CALCULATION PROCEDURE

To calculate the operating temperatures in the different layers of the inductor, we consider the algorithm of figure 10. This algorithm summarizes the different steps.

Figure 10. Procedure for calculating the temperatures in the different inductor layers

4.1 Thermal resistances calculation

By analogy between the physical laws of thermal conduction and electric conduction, the thermal resistances are expressed by equation (1) [13] [14].

$$R_{th} = \frac{d}{\lambda_{th} \cdot S_k} \quad (1)$$

d : is the distance between the node and the interface.

λ_{th} : is the thermal conductivity of the material.

S_k : is the section crossed perpendicularly by the thermal flow.

The conductive ribbon of the inductor is sectioned into volumes $V_{l,k}$ ($k=1, \dots, 9$, each volume $V_{l,k}$ is a heat source. The heat flow transferred from each volume $V_{l,k}$ to the layers stacked below, crosses perpendicularly the section S_k . The calculation of the thermal resistances is done after the calculation of the sections S_k ($k=1, \dots, 9$) (Table II)

TABLE II
EQUATIONS AND VALUES OF SECTIONS S_k

Section S_k ($k = 1, \dots, 9$)	Equations	Results (μm^2)
S_1	$S_1 = (\frac{d_{in}}{2} + 2w + s).w$	54000
S_2	$S_2 = (d_{in} + 3w + 3s).w$	105300
S_3	$S_3 = (d_{in} + 4w + 2s).w$	108000
S_4	$S_4 = (d_{in} + 2w + 2s).w$	97200
S_5	$S_5 = (d_{in} + 3w + s).w$	81900
S_6	$S_6 = (d_{in} + w + s).w$	53100
S_7	$S_7 = (d_{in} + 2w).w$	55800
S_8	$S_8 = d_{in}.w$	27000
S_9	$S_9 = (\frac{d_{in}}{2} + w).w$	27900

4.1.1 The thermal resistances of different materials

The thermal resistances of the different stacked materials: copper (Cu), silica (SiO₂), permalloy (NiFe) and silicon substrate (Si) are respectively R_{thc_k} , R_{thox_k} , R_{thp_k} , R_{thsi_k} .

The respective thermal conductivities values of the stacked materials are:

$$\lambda_{cu}=349 \text{ Wm}^{-1}\text{K}^{-1}, \lambda_{ox}=1,46 \text{ Wm}^{-1}\text{K}^{-1}, \lambda_{mag}=30 \text{ Wm}^{-1}\text{K}^{-1}, \lambda_{sub}=149 \text{ Wm}^{-1}\text{K}^{-1}$$

The calculation of the thermal resistances is done using equations (2).

$$R_{thc_k} = \frac{d_{cu}}{S_k \cdot \lambda_{cu}}, R_{thox_k} = \frac{t_{ox}}{S_k \cdot \lambda_{ox}}, R_{thp_k} = \frac{h_{mag}}{S_k \cdot \lambda_{mag}}, R_{thsi_k} = \frac{h_{sub}}{S_k \cdot \lambda_{sub}} \quad (2)$$

d_{cu} : Is the distance between the node of volume $V_{l,k}$ and the silica-copper interface ($d_{cu}=10\mu m$).
 t_{ox} , h_{mag} and h_{sub} Are the respective thicknesses of the different stacked materials (silica , permalloy and silicon), their values are respectively $t_{ox}=60\mu m$, $h_{mag}=340\mu m$ and $h_{sub}=100\mu m$.
Table III contains the calculated values of the thermal resistances of different materials in each volume $V_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$).

TABLE III
THERMAL RESISTANCES OF DIFFERENT STACKED MATERIALS

$R_{thc_k} (K/W)$	$R_{thp_k} (K/W)$	$R_{ths_k} (K/W)$	$R_{thsi_k} (K/W)$
0.476	209.88	761.03	12.43
0.244	107.63	390.27	6.37
0.238	104.95	380.52	6.21
0.264	116.60	422.79	6.90
0.314	138.38	501.78	8,19
0.484	213.43	773.93	12.64
0.461	203.11	736.48	12.03
0.952	419.75	1522.20	24.86
0.921	406.21	1472.97	24.06

5 CALCULATION OF THE INDUCTOR OPERATING TEMPERATURE

The calculation of the temperatures in the different volumes $V_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) of the inductor (Figure 11) is done with the use of Fourier's law given by equation (3).

$$R_{th} = \frac{\Delta T}{\Phi} = \frac{T_i - T_j}{\Phi} \quad (3)$$

T_i and T_j are the temperatures of two isotherms, Φ is the flow of heat between the two isotherms expressed in Watts. By applying the Fourier's law to Figure 11, we obtain equation (4).

$$R_{i,k} = \frac{T_{i,k} - T_{i+1,k}}{\Phi_{1,k}} \quad (4)$$

$R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) are the thermal resistances between two nodes $A_{i,k}$ and $A_{i+1,k}$

$\Phi_{1,k}$ ($k=1, \dots, 9$) are the Partial thermal flows in the volumes $V_{1,k}$ ($k=1, \dots, 9$).

To calculate temperatures $T_{i,k}$ in each volume $V_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$), it is first necessary to calculate the resistances $R_{i,k}$ and the Partial thermal flows $\Phi_{1,k}$.

The thermal flows $\Phi_{1,k}$ derive from the overall thermal flow Φ_{total} .

Figure 11. The different thermal resistances and different temperatures in a volume element V_k

5.1 Calculation of the overall thermal flow

The overall thermal flow Φ_{total} due to losses by Joule effect is assumed uniform throughout the conductor. This thermal flow represents the power dissipated by the conductive ribbon (equation 5).

$$\Phi_{total} = P = R_S \cdot (I_{Leff})^2 \quad (5)$$

R_S is the ohmic resistance of the conductive ribbon given by equation (6)

$$R_S = \rho_{cu} \frac{L_{avg}}{w \cdot t} \quad (6)$$

ρ_{cu} is the resistivity of copper $\rho_{cu} = 1.7 \cdot 10^{-8} \Omega m$

With $I_{Leff} = 0.4A$ and $R_S = 0.0311 \Omega$ we obtain:

$$\Phi_{total} = 4,976 \cdot 10^{-3} W \quad .$$

5.2 Calculation of partial thermal flows

The conductive ribbon of the inductor is discretized into nine volumes $V_{i,k}$ ($i=1, \dots, 4; k=1, \dots, 9$), their temperatures $T_{i,k}$ ($k=1, \dots, 9$) are assumed isothermal, the power dissipated by each volume $V_{i,k}$ ($k=1, \dots, 9$) represents a partial thermal flow $\Phi_{i,k}$ ($k=1, \dots, 9$). The partial thermal flows derive from the overall thermal flow Φ_{total} (expression 7).

$$\Phi_{i,k} = \frac{V_{i,k}}{V_g} \Phi_{total} \quad (7)$$

V_g is the overall volume of the conductive ribbon.

Table IV shows the values of different thermal flows $\Phi_{i,k}$ ($k=1, \dots, 9$) in volumes $V_{i,k}$ ($k=1, \dots, 9$).

TABLE IV
VALUES OF OVERALL AND PARTIAL THERMAL FLOWS IN THE VOLUMES $V_{1,k}$

Volumes V_g and $V_{1,k}$	Values of volumes $V_{1,k}$ (μm^3)	Thermal flows	Values of thermal flows $\Phi_{1,k}$ (W)
Averall volume $V_g = L_{avg} \cdot t \cdot w$	10566.10 ³	Φ_{total}	4.976.10 ⁻³
$V_{1,1}$	108.10 ⁻⁴	$\Phi_{1,1}$	0.5088.10 ⁻³
$V_{1,2}$	2106.10 ⁻³	$\Phi_{1,2}$	0.9916.10 ⁻³
$V_{1,3}$	216.10 ⁻³	$\Phi_{1,3}$	1.0172.10 ⁻³
$V_{1,4}$	1944.10 ⁻³	$\Phi_{1,4}$	0.9156.10 ⁻³
$V_{1,5}$	1638.10 ⁻³	$\Phi_{1,5}$	0.7712.10 ⁻³
$V_{1,6}$	1062.10 ⁻³	$\Phi_{1,6}$	0.5.10 ⁻³
$V_{1,7}$	1116. .10 ⁻³	$\Phi_{1,7}$	0.5256.10 ⁻³
$V_{1,8}$	54.10 ⁻³	$\Phi_{1,8}$	0.2544.10 ⁻³
$V_{1,9}$	558.10 ⁻³	$\Phi_{1,9}$	0.2628.10 ⁻³

5.3 Calculation of thermal resistances between nodes $A_{i,k}$ and $A_{i+1,k}$

Figure 12 is deduced from figure 11.

In figure 12, $R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) are the thermal resistances between the nodes $A_{i,k}$ and $A_{i+1,k}$. $A_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) are the centers of gravity of volumes $V_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) (Figure 11), and resistances $R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) are the elements of a matrix (Figure 12).

Figure 12. Different thermal resistances

The calculation of the thermal resistances is done using the expressions (6).

$$R_{1,k} = R_{thc_k} + \frac{R_{ths_k}}{2}, \quad R_{2,k} = \frac{R_{ths_k} + R_{thp_k}}{2}, \quad R_{3,k} = \frac{R_{thp_k} + R_{thsi_k}}{2}, \quad R_{4,k} = \frac{R_{thsi_k}}{2} \quad (6)$$

The matrix of resistances $R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$) in table V derived from figure 12.

TABLE V
MATRIX OF THERMAL RESISTANCES $R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$).

$$\begin{bmatrix} R_{1,1} & R_{1,2} & R_{1,3} & R_{1,4} & R_{1,5} & R_{1,6} & R_{1,7} & R_{1,8} & R_{1,9} \\ R_{2,1} & R_{2,2} & R_{2,3} & R_{2,4} & R_{2,5} & R_{2,6} & R_{2,7} & R_{2,8} & R_{2,9} \\ R_{3,1} & R_{3,2} & R_{3,3} & R_{3,4} & R_{3,5} & R_{3,6} & R_{3,7} & R_{3,8} & R_{3,9} \\ R_{4,1} & R_{4,2} & R_{4,3} & R_{4,4} & R_{4,5} & R_{4,6} & R_{4,7} & R_{4,8} & R_{4,9} \end{bmatrix}$$

Table VI shows the calculated values of the different thermal resistances $R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$).

TABLE VI
VALUES OF THERMAL RESISTANCES $R_{i,k}$ ($i=1, \dots, 4$; $k=1, \dots, 9$)

$$\begin{bmatrix} 380,99 & 195,38 & 190,50 & 211,66 & 251,20 & 387,45 & 368,70 & 762,05 & 737,41 \\ 485,46 & 248,95 & 242,74 & 269,70 & 320,08 & 493,68 & 469,80 & 970,98 & 939,59 \\ 111,16 & 57,00 & 55,58 & 61,75 & 73,29 & 113,06 & 107,57 & 222,31 & 215,16 \\ 06,22 & 03,19 & 03,11 & 03,45 & 04,10 & 06,32 & 06,02 & 12,43 & 12,03 \end{bmatrix}$$

5.4 Calculation of the operating temperatures of the inductor

To calculate the operating temperature $T_{i,k}$ at each node $A_{i,k}$, we use expression 8.

$$R_{i,k} = \frac{\Delta T}{\Phi} = \frac{T_{i,k} - T_{i+1,k}}{\Phi_{1,k}} \Rightarrow T_{i,k} - T_{i+1,k} = R_{i,k} \cdot \Phi_{1,k} \quad (8)$$

By setting k and varying i from 1 to 4 in expression (8), we obtain the equations system (9).

$$\begin{cases} T_{1,k} - T_{2,k} = R_{1,k} \cdot \Phi_{1,k} \\ T_{2,k} - T_{3,k} = R_{2,k} \cdot \Phi_{1,k} \\ T_{3,k} - T_{4,k} = R_{3,k} \cdot \Phi_{1,k} \\ T_{4,k} - T_{5,k} = R_{4,k} \cdot \Phi_{1,k} \end{cases} \Rightarrow \begin{cases} T_{1,k} = R_{1,k} \cdot \Phi_{1,k} + T_{2,k} \\ T_{2,k} = R_{2,k} \cdot \Phi_{1,k} + T_{3,k} \\ T_{3,k} = R_{3,k} \cdot \Phi_{1,k} + T_{4,k} \\ T_{4,k} = R_{4,k} \cdot \Phi_{1,k} + T_{5,k} \end{cases} \quad (9)$$

With k varying from 1 to 9, we obtain nine similar equations systems.

We assume that the ground plane is at ambient temperature (20 °C), therefore $T_{5,k}=20^\circ\text{C}$ for ($i=1, \dots, 4$), and we solve the equations system (9).

The results are given in table VII for ($k=1, \dots, 9$).

TABLE VII
OPERATING TEMPERATURES IN DIFFERENT AREAS OF THE INDUCTOR

$T_{Cu} \rightarrow$	20,504	20,503	20,504	20,503	20,504	20,504	20,504	20,504	20,504
$T_{SiO_2} \rightarrow$	20,31	20,31	20,31	20,31	20,31	20,31	20,31	20,31	20,31
$T_{NiFe} \rightarrow$	20,06	20,06	20,06	20,06	20,06	20,06	20,06	20,06	20,06
$T_{Si} \rightarrow$	20,003	20,003	20,003	20,003	20,003	20,003	20,003	20,003	20,003

5.5 Results interpretation

Since the operating temperature of the electronic power devices is between 20°C and 120°C [9], we conclude that the results in Table VII are good.

The temperature of the permalloy layer (NiFe) is 20.06 ° C, so there is no risk of demagnetization of the core because the Curie temperature of the permalloy (600 ° C.) is far from being reached.

We note also that the temperature of the silicon layer is uniformly distributed and is not high, therefore, there is no risk of increasing the intrinsic conductivity of the semiconductor and of the conductivity of the substrate.

To see the heat distribution in the conductive ribbon of the inductor during its operating, a simulation of the temperature was performed with the FEMLAB 3.1 software (Figure 13-14-15).

Figure 13. Simulation of the thermal flow in the inductor before its dissipation

The thermal flow is uniformly distributed along the conductive ribbon. Figure 13 shows no skin effect, no proximity effect. We conclude therefore, it has no risk of overheating the conductive ribbon.

In order to simulate the heat transfer, we place the conductive tape on the insulating layer and make cross-sections of these two layers (figure 14 and figure 15).

Figure 14. Conductive ribbon laid on the insulating layer

Figure 15. Transfer of the thermal flow from the conductive ribbon toward the insulating layer

Figure 15 shows the heat transfer along the conductive ribbon.

During the operation of the inductor, the heat does not stagnate in the conductive ribbon, but it is transferred to the stacked layers below to be evacuated outwardly through the silicon layer.

6 CONCLUSION

The integrated inductor is a fundamental element in a micro-converter, therefore the study of the thermal behavior of the inductor is very important.

To describe the thermal behavior of the inductor and calculate its operating temperature, we proposed a simple approach. This approach is based on the design of the inductor's thermal circuit. The nodal method, the cross section of the inductance and the similarity between the thermal and electrical circuits, facilitated the design of a " π " thermal circuit of the inductor.

Fourier's law and the established thermal circuit, allowed us to determine the operating temperatures in different areas of the inductor. The calculated temperatures are good, so, the results are encouraging.

We validate this work using the FEMLAB 3.1 software. The image provided from the simulation shows no skin effect and no proximity effect, therefore, the operating temperature are uniform and there is no risk of overheating of the conductive ribbon or the inductor.

Heat does not stagnate in the conductive ribbon, but it is transferred to the stacked layers below, to be evacuated to the external environment through the silicon substrate which is a good heat conductor.

We conclude so, that the approach presented in this paper is reliable, simple and very easy to applied, compared to other methods such as the finite element method and the finite difference method.

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