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Design and Analysis of a Magnetically Geared Induction Machine

B. Bidouche, T. Lubin, S. Mezani

Abstract -- This paper describes the electromagnetic and thermal design of a magnetically geared induction motor (MaGIM). The device consists of a wound-rotor induction machine artfully coupled to a magnetic gear to achieve a high torque density low speed drive system. The aim is to replace the conventional system which combines a high-speed induction motor and a mechanical gear as to improve performance and reliability.

Index Terms—Induction motor, magnetic gear, finite elements, analytical model.

I. INTRODUCTION

The idea of a magnetic gear with permanent magnets started in 1940 when Faus patented an original magnetic gear topology [1]. Then more works have been done to achieve higher torque density. In the last decade, high performance magnetic gears have been proposed with several ways of permanent magnets arrangements [2], [3], [4], [5]. Magnetic gears have also been associated to PM synchronous machines to achieve a high torque density low-speed drive system [6]-[7].

Despite these recent advance, there are numbers of industrial applications where it is well suited to employ an induction motor which present the advantage of self-starting when connected to the mains. Hence, a new concept called magnetically geared induction machine (MaGIM) was introduced in [8]. It associates an induction machine with a magnetic gear to achieve a high-torque/low-speed drive system with extended torque transmission capabilities. The aim is to improve the reliability and the performances of the traditional system associating a high-speed induction machine (IM) with a mechanical gearbox which exhibit mechanical problems and requires lubrication and maintenance.

We propose to study a wound rotor induction motor drive integrating a magnetic gear, which constitutes the MaGIM, the machine being supplied directly via the mains. We present in this paper the design of a 1 kW – 45 rpm MaGIM operating on the 400 V / 50 Hz network. We use for this purpose analytical and numerical computation tools to predict the electromagnetic and thermal performances of the

device.

This paper is divided into three further sections, where Section II deals with a basic description of the Magnetically Geared Induction Machine. Next, in Section III, the development of a design procedure of the MaGIM is described. In Section IV, the performances of the machine are presented and, finally, a conclusion is given in Section V.

II. PRINCIPLE OF OPERATION OF THE MAGIM

As shown in Fig. 1, the studied system consists of a magnetic gear coupled to a reverse type induction motor having an inner stator and an outer rotor. The motor represents the internal part of the MaGIM while the magnetic gear represents the external part. The high-speed rotor is the common part of the motor and the magnetic gear. The conventional magnetic gear involves two rotors and a fixed armature.

The inner high-speed rotor has p_i pole-pairs radially magnetized permanent magnets. The low speed rotor consists of P_s ferromagnetic pole-pieces and the stationary outer armature has p_o pole-pairs radially magnetized permanent magnets.

The principle of operation of the magnetic gear is based on the modulation of the magnetic field created by the p_i PM pole-pairs by the P_s ferromagnetic pole pieces.

Fig. 1. 2D view of the MaGIM.

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The obtained field interacts with the p_o PM pole-pairs of the external armature to transmit torque to the low-speed load. The combination $p_o = P_s - p_i$ results in the highest torque transmission capability of the gear [2].

The wound rotor induction motor has p pole-pairs three-phase windings in the stator and the rotor armatures. This wound rotor configuration leads to a high starting torque obtained by adding resistances in series with the rotor winding. Indeed, we are looking here for a MaGIM system directly supplied via the mains for fixed speed application requiring high starting torque. The electromagnetic torque production in the IM moves the high-speed rotor at a mechanical speed ω_i . The external armature being held stationary, the torque is transmitted to the low-speed load via the pole-pieces. This torque corresponds to the one imposed by the load at a speed ω_s . It is important to note that the net average torque on the high-speed rotor (motor + magnetic gear) is equal to zero. The speeds ω_i and ω_s are related by [2]:

$$\omega_i = \frac{P_s}{P_s - p_o} \omega_s \quad (1)$$

Thus, the resulting gear ratio G_r is:

$$G_r = \frac{P_s}{p_i} \quad (2)$$

III. DESIGN PROCEDURE OF THE MaGIM

The design of the MaGIM is complex owing the large number of geometric and physical parameters to be considered. Indeed, there are many electromagnetic interactions between the different parts of the device, which makes it difficult to do a design with a simple sizing procedure. Furthermore, the thermal aspects are important, so a simplified thermal model is also developed to achieve a more precise design of MaGIM.

A. Electromagnetic sizing of MaGIM

The design process of the MaGIM requires the analysis of an asynchronous machine and a synchronous system (the magnetic gear). Here, we choose to study the two sub-systems separately.

To achieve the electromagnetic sizing of MaGIM, we have used analytical like models for preliminary computations and finite elements to get more precise results.

- The machine air-gap diameter and active length have been firstly calculated using the electrical and magnetic loading concept together with past experience from machine manufacturers [10]. The heating factor (the product of the electric loading and the current density) has been set to a maximum value of $1000 \text{ A}^2/\text{mm}^2.\text{cm}$ which leads to minimum cooling requirements for the machine. Then, after choosing the number of slots, one can determine the dimensions of the teeth, the back-iron, the slots and the number of turns per phase. A static linear FE computation

has then been used to compute the equivalent circuit parameters of the IM from which all the machine performances are determined. Finally, non-linear transient FE computation allows a more precise calculation of the IM.

- The magnetic gear sizing uses a 2D analytical model based on the sub-domain technique [12]. Then, non-linear FE computation allows a more precise calculation of the gear performances.

Many simulations are needed to achieve a good compromise between the optimal performances of the machine and the gear sides. In the final stage, finite element analyses are carried out to consider the overall system.

We seek to design the most compact device (high torque density of the gear) with the best possible energetic performances (IM with high efficiency, high power factor...). We need then to make a compromise between the performances of the induction motor and the magnetic gear.

The load torque is exerted on the pole-pieces. For a MaGIM rated at 1 kW - 45 rpm, the torque on the pole pieces is about 210 Nm. In our study, the design is done for a torque of 350 Nm as to consider a safety margin. This margin includes 3D effects not taken into account in the 2D computation models but also to allow for any transient operation which requires torque values larger than the rated one (starting, over-load...).

For a 1 kW IM, the slip s is usually around 0.06 to 0.1. Hence for $s=0.1$ and a supply frequency of 50 Hz, the gear ratio is $G_r=60,30,20,15,12,10$ for $p=1,2,3,4,5,6$ respectively. Induction motors have better performances for low p values [10] while the magnetic gear has its higher torque density for a gear ratio G_r ranging from 5 to 8 [11]. Then, the compromise adopted here to achieve acceptable energetic performances and compactness consists of choosing $p=4$ for the IM. For the magnetic gear, we have adopted the combination $p_i=2$, $p_o=29$, $P_s=31$ which leads to a gear ratio $G_r = 15.5$ and minimal torque ripples [9]. This choice is also made to get reasonable dimensions of the pole-pieces as to insure their mechanical integrity while transmitting the torque.

The electromagnetic design led to the active dimensions of the MaGIM indicated in Table 1. With these dimensions, the gear exhibits a pull-out torque exerted on the pole-pieces of 350 Nm. The rated torque of the IM is equal to 14 Nm obtained for a slip value of 0.08.

B. Thermal model of MaGIM

A simplified steady state thermal network of the MaGIM has been constructed to determine the temperature distribution in the active parts. A radial heat flux is considered except for the slots for which the ortho-radial heat flux to the teeth is accounted for. The model is based on lumped parameters which involve the computation of thermal resistances [13], [14], [15]. In the solid parts, conductive thermal resistances are calculated from the

thermal conductivities and the dimensions. Note that bulk thermal conductivities are used in the slots.

TABLE 1
MAIN GEOMETRICAL PARAMETERS OF MAGIM

Magnetic gear side	
Inner rotor pole-pairs p_i	2
Stationary armature poles-pairs p_o	29
Number of pole-pieces P_s	31
Gear ratio G_r	15.5
Airgap (inner and outer rotor sides)	2 mm
Remanence of PMs B_r (NdFeB)	1.2 T
External radius	131 mm
Inner rotor radius	80.5 mm
Active length	150 mm
Induction machine side	
Stator rated line voltage (Y connexion)	400 V
Supply frequency	50 Hz
Pole pairs p	4
Active length	150 mm
Stator outer radius	61 mm
Airgap	0.3 mm
Number of stator slots	24
Number of rotor slots	48
Stator turns per phase	296
Rotor turns per phase	160

The convection heat transfer concerns the air-gaps, the external surface of MaGIM and the inner hole of the stator. The heat transfer coefficients are computed using well known formulae [13], [14], [15]. The stator is mainly cooled through the inner hole in which a fan blows an air flow. Table 2 gives the values of the different thermal coefficients used in the thermal model.

The convection coefficient h_{hole} of the inner stator hole is calculated by:

$$h_{hole} = \frac{Nu_{is} \cdot k}{R_{is}} \quad (3)$$

Where R_{is} represent the inner stator radius and k_{air} the thermal conductivity of air. Nu_{is} represents the Nusselt coefficient whose value for an internal forced convection is related to the Reynolds Re and Prandtl Pr numbers by:

$$Nu_{is} = A \cdot Re^n \cdot Pr^m \quad (4)$$

The values of A , n and m are given by:

$$\begin{cases} A = 3.66, n = 0, m = 0 & \text{if } Re < 2000 \\ A = 0.023, n = 0.8, m = 0.33 & \text{if } Re > 2000 \end{cases} \quad (5)$$

The convection coefficients for the three airgap regions (IM, low and high speed gear sides) are expressed by:

$$h_{gap} = \frac{Nu_g \cdot k_{air}}{e_{gap}} \quad (6)$$

Where h_{gap} is the convection coefficient of the considered airgap region and e_{gap} the airgap thickness.

The Nusselt number Nu_g is related to Taylor number Ta by:

$$\begin{cases} Nu = 2.2 & \text{if } Ta < 41 \\ Nu = 1.116 \cdot Ta^{0.63} \cdot Pr^{0.27} & \text{if } Ta > 41 \end{cases} \quad (7)$$

Figure 2 shows the thermal network of MaGIM which consists of 15 nodes. The active parts are numbered as follows:

1(stator back-iron), 3(stator slot), 4(stator tooth), 7(high-speed rotor slot), 8(high-speed rotor tooth), 10(high-speed rotor back-iron), 11(high-speed rotor PMs), 14(stationary armature PMs), 15(stationary armature back-iron).

The heat sources (Joule and iron losses) are injected to these active nodes. They are obtained from the electromagnetic model presented above. Note that the low speed rotor pole-pieces are made from SMC materials whose losses are neglected. The pole-pieces are inserted in a fiber glass container to form the low-speed rotor. In this way, a sufficient structural strength to transmit the torque is ensured.

Fig. 2. Equivalent thermal network of MaGIM

TABLE 2
THERMAL COEFFICIENTS USED IN THE THERMAL NETWORK

Region	Convection coefficient [W/m ² .°C]
Inner stator hole	98.7
Airgap (IM side)	220
Airgap (high-speed rotor gear side)	33
Airgap (stationary armature gear side)	33
External surface of fixed armature	50
Region	Thermal conductivity [W/m.°C]
Iron laminations	28
Slot winding	1
NdFeB PMs	8.5
SMC pole-pieces	2.2
fiberglass	0.02

IV. RESULTS AND DISCUSSION

A. Electromagnetic performances

As stated above, a per-phase equivalent circuit is used to compute the performances of the induction motor. Table 3 gives the values of the electrical parameters of the machine. Note that the 2D FE model ignores the end-winding leakage inductances. Usual formulae [10] are used to compute these inductances and their values added to the ones computed in 2D. To get more precise results, a time-harmonic (TH) non-linear (NL) 2D FE simulation with circuit coupling has been performed. The B-H curve of M400-50A laminations is used for the ferromagnetic parts.

For slip values up to 0.1, Fig. 3 shows the torque vs. slip curves computed by the two models. The agreement is very good, the relative difference doesn't exceed 2%. The rated torque of 14 Nm is reached for a slip value of 0.08.

Fig. 4 shows the stator current vs. slip curves obtained using the two models. The results issued from the equivalent circuit underestimate the no-load current. The relative difference is about 8% for $s=0$. For higher slip values ($s > 0.04$), the agreement is good as the relative difference doesn't exceed 3%. The rated current obtained at a slip of 0.08 is about 3.1 A.

Fig. 5 shows the power factor vs. slip curves obtained using the two models. Again, the agreement is very good as the relative difference doesn't exceed 3%. The rated PF is around 0.58 which is relatively low. However, this represents usual FP values for small power induction motors with high pole numbers.

Fig. 6 shows the static torque vs. position of the low-speed rotor; the high-speed rotor being held stationary. Both 2D and 3D FE computations are performed. The torque curve is almost sinusoidal and ripple free. The 2D pull-out torque is about 350 Nm while the 3D model gives 310 Nm. Hence, to obtain the right torque value while still using a 2D model, we simply multiply the 2D torque by a correction factor of about 0.88 (ratio of the 3D and 2D pull-out torques).

TABLE 3

EQUIVALENT CIRCUIT PARAMETERS OF THE INDUCTION MOTOR

Electrical parameter	Value
Stator inductance L_s	0.276 [H]
Rotor inductance L_r	0.065 [H]
Mutual inductance M	0.121 [H]
Stator resistance R_s	6.3 [Ω]
Rotor resistance R_r	2.06 [Ω]

Fig. 3. Electromagnetic torque vs. slip curves

Fig. 4. Stator current vs. slip curves

Fig. 5. Power factor vs. slip curves

Fig. 6. Static torque vs. angular position of the low-speed rotor

For the rated operating conditions at constant speeds (45 rpm for the low-speed rotor and 680 rpm for the high-speed rotor), a non-linear time-stepping 2D FE simulation with movement consideration and circuit coupling has been performed for the whole system. The relative positions between the high and the low speed rotors is set to get a corrected 2D torque value of 212 Nm on the low speed rotor.

Fig. 7 shows the dynamic torque waveform of the low speed rotor. This torque is practically ripple free with a mean value of 212 Nm; the torque ripples don't exceed 1%.

Fig. 8 presents the total torque exerted on the high-speed rotor. Of course, the mean torque is equal to zero since the gear and the machine develop opposite torques whose mean values are about 14 Nm. However, one can observe the torque ripples due to the machine and gear sides, these ripples represent 26% of the rated torque of 14 Nm. This is one of the great advantages of the magnetic gear which can transmit a free-ripple torque to the load (low speed) while the input torque (high speed) contains high pulsations.

Fig. 7. Low-speed rotor dynamic torque at rated operation

Fig. 8. High-speed rotor dynamic torque at rated operation

The active torque density at rated operation of MaGIM is about 26 Nm/L. A mechanical gear with the same gear ratio exhibits a torque density of about 15 Nm/L [11]. When associated to a high-speed induction motor having a standard torque density of 10 Nm/L, the overall system (IM + mechanical gear) will exhibit a torque density of 14 Nm/L. This clearly shows that MaGIM is superior to the conventional system with the benefit of better reliability and low maintenance requirements.

B. Performances of the thermal model

The steady state temperature distribution in the active parts of the machine are computed for rated load conditions. The different losses computed by the electromagnetic model are given in Table 4. Notice that the iron losses are evaluated by the standard formulae [10]. Furthermore, the losses in the gear side (magnets and back-iron) are very low for the studied MaGIM so their values are set to zero for the thermal computation.

Table 5 gives the computed temperatures in the different regions of the device. As expected, the stator slots reach the highest temperature of 107°C. The high-speed rotor temperatures vary between 91 and 98°C, the lowest one concerns the PMs whose temperature reaches 91°C.

The low speed rotor and the external armature temperatures are about 78°C and 66°C respectively because the large air-gaps on the gear sides act as a heat flux barrier. It is important to give a special care the high-speed rotor PMs as a high operating temperature could result in demagnetization problems. Hence, it is probably safer to use SmCo magnets on the high-speed rotor although they are more expensive than NdFeB PMs.

Nevertheless, the simplified 2D thermal model ignores the axial heat transfer, so the computed temperatures are probably higher compared the ones we would obtain if the axial heat transfer is considered.

TABLE 4
DIFFERENT LOSSES IN MAGIM AT RATED OPERATION

Region name	Losses [W]
Stator back-iron (node 1)	18
Stator tooth (node 4)	32
Stator slots Joule losses (node 3)	194
Rotor slots Joule losses (node 7)	131

TABLE 5
TEMPERATURE DISTRIBUTION IN MAGIM AT RATED OPERATION

Region name	Temperature, T[°C]
Stator slots (node 3)	106.92
Stator teeth (node 4)	96.94
Stator back- iron (node 1)	93.85
High-speed rotor slots (node 7)	98.36
High-speed rotor teeth (node 8)	92.98
High-speed rotor back-iron (node 10)	91.67
High-speed rotor PMs (node 11)	91.58
Low-speed rotor (node 12)	78.01
Fixed armature PMs (node 14)	65.81

V. CONCLUSION

In this paper, the sizing procedure of a magnetically geared induction machine MaGIM rated at 1 kW - 45 rpm has been presented. The device is intended for a constant speed operation on the 400 V-50 Hz network. Both electromagnetic and thermal analyses have been carried out by means of analytical and finite element computation tools.

We have shown that a compromise must be made to reach a compact magnetic gear while keeping acceptable energetic performances of the induction motor.

To estimate the temperature of MaGIM, we built a simplified lumped-parameter thermal model. The model was formulated from physical constants, heat transfer characteristics and dimensional data. It has been shown that the high-speed PMs can reach a relatively high temperature so the use of SmCo magnets would be preferable.

The designed MaGIM is currently under construction and we hope to publish the first experimental results in few months.

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VII. BIOGRAPHIES

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