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Study of the thrust–drag balance with a swimming robotic fish

Florence Gibouin,¹ Christophe Raufaste,¹ Yann Bouret,¹ and Médéric Argentina^{1,2,a)}

¹Université Côte d'Azur, CNRS, Institut de Physique de Nice, 06100 Nice, France

²Institut Universitaire de France, 75005 Paris, France

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A robotic fish is used to test the validity of a simplification made in the context of fish locomotion. With this artificial aquatic swimmer, we verify that the momentum equation results from a simple balance between a thrust and a drag that can be treated independently in the small amplitude regime. The thrust produced by the flexible robot is proportional to $A^2 f^2$, where A and f are the respective tail-beat amplitude and oscillation frequency, irrespective of whether or not f coincides with the resonant frequency of the fish. The drag is proportional to U_0^2 , where U_0 is the swimming velocity. These three physical quantities set the value of the Strouhal number in this regime. For larger amplitudes, we found that the drag coefficient is not constant but increases quadratically with the fin amplitude. As a consequence, the achieved locomotion velocity decreases, or the Strouhal number increases, as a function of the fin amplitude. Published by AIP Publishing. <https://doi.org/10.1063/1.5043137>

I. INTRODUCTION

Most macroscopic swimmers undulate their bodies to generate a net thrust and propel themselves. Using Theodorsen's aerodynamics of rigid wings,¹ in 1936, Garrick studied self-propelled vibrating foils.² Since then, flapping foils have been efficient experimental devices to model fish propulsive motion.^{3–5} This approach has resulted in showing that an optimum for propulsion is achieved through a combination of heave and pitch with a phase difference of 90° .^{6,7}

Concentrating on the theoretical aspects of fish propulsion, Lighthill established the slender-body theory for deformable bodies to make thrust calculations.^{8,9} The effect of appendage flexibility has since been investigated in pitching movements,^{10,11} heaving motions,^{12–16} and a mix of them both^{17,18} or by confining the swimmer in the vicinity of a wall.¹⁹ For all the studies mentioned, the thrust is notably enhanced when the foils are forced near their resonant frequencies.²⁰ These are associated with the eigenmodes of the system comprised of the plate-like foil coupled to its surrounding fluid. For elastic plates forced by an oscillating torque distribution, the highest velocities are also observed near the resonant frequency forcing.²¹

Data collected from natural swimmers show that the swimming characteristics in the turbulent regime, i.e., a Reynolds number Re larger than 3000,²² are governed by the relative uniformity of two dimensionless numbers, the Strouhal number $St = Af/U_0 \sim 0.3$ ^{22–27} and the ratio $A/L \sim 0.2$,^{26–29} with swimming velocity U_0 , fish length L , tail-beat amplitude A , and frequency f . To understand these data in terms of swimming efficiency, numerous Refs. 4–7, 10, and 30–34 have assumed that the mean thrust F_T scales with the dynamic pressure and that the thrust coefficient should be defined as $F_T / \frac{1}{2} \rho S_T U_0^2$, where ρ is the fluid density and

S_T is the effective propulsive area of the fish. Dewey *et al.*¹¹ emphasized that such a definition is not appropriate when accounting for flexible fins and their resonant behavior.

Recently, Gazzola *et al.* brought new insights by analyzing data of natural swimmers over nearly eight orders of magnitude of the Reynolds number.²² They had shown that the relative uniformity of the Strouhal number in the turbulent regime results from a simple thrust–drag balance, with thrust scaling as $\rho L^2 A^2 f^2$ and frictional drag scaling as $\rho L^2 U_0^2$. They also reported that in the laminar regime (Reynolds number smaller than 3000), the relevant dimensionless parameter changes due to a change in the drag expression to account for the viscous boundary layer. Actually a subtle hypothesis is hidden behind this simple balance:²¹ to the leading order, thrust and drag can be treated separately, with thrust mostly independent of the swimming velocity^{11,21,35} and drag mostly independent of the fin frequency or amplitude.²¹ It is remarkable that this idea was already in the mind of Lighthill⁸ and that his definition of the thrust coefficient C_T seems to be the natural one,^{9,36}

$$C_T = F_T / \frac{1}{2} \rho S_T A^2 f^2. \quad (1)$$

This emphasizes that the thrust expression should not depend on the fluid velocity nor directly on the system's flexibility in the range of parameters found in natural fish.

In this manuscript, we use a swimming robot with a flexible tail, flexible body, and rigid head to:

- Perform force measurements to test the simple thrust–drag balance assumption. Thrust and drag are first considered separately and then together to test whether the swimming characteristics can be inferred from an *a priori* separate treatment.
- Find out whether for the dynamics of flexible structures, which are known to be efficient close to their resonant frequencies, the thrust scales as $A^2 f^2$, as already seen for rigid panels.^{11,14,35,37}

^{a)} Author to whom correspondence should be addressed: mederic.argentina@unice.fr

- Test a flexible design that is slightly more advanced than that of flexible and elastic panels. The driving torque is thus distributed all along the fish body instead of being localized at the leading edge.

The manuscript is organized as follows. In Sec. II, we describe the experimental setup, including the fabrication and control of the robot, force measurements, and image analysis. The three main control parameters are the free stream velocity U_0 and the two fish motion parameters: the imposed servomotor amplitude Φ and oscillation frequency f .

In Sec. III, thrust and drag are studied separately. Both the tail-beat amplitude and thrust are measured as functions of Φ and f without any free stream flow ($U_0 = 0$). The thrust is found to scale as $A^2 f^2$, while A is maximized for the given values of the imposed parameters. This reconciles the thrust expression with the resonant behavior observed with flexible panels¹⁶ since a resonant fin triggers a higher amplitude of oscillation and thus a higher thrust. Several caudal fin designs were also tested to validate the thrust expression. The drag is measured separately as a function of the free stream velocity ($U_0 \neq 0$) for several positions of the fin while the fish is at rest. We shed light on a correction factor to account for an enhanced drag when the fin is positioned off-center. Finally, to mimic real swimming conditions, U_0 , Φ , and f are varied systematically. Self-propulsion at constant speed is achieved when an exact balance between thrust and drag occurs. The values of the parameters (A , f , and U_0) giving rise to undulatory locomotion are characterized experimentally and analyzed in Sec. III C by considering the previous measurements of the thrust and drag made separately. Further discussion is presented in Sec. IV.

II. MATERIAL AND METHODS

Following the work of Triantafyllou,³⁸ various compliant robotic devices have been used to assess the mechanics at play in swimming locomotion.^{17,39–44} Here, we propose our own robot. The experimental setup is comprised of a robotic fish with a flexible tail, flexible body, and rigid head immersed in a water tunnel (Rolling Hills Research Corporation, Model 0710). The test section is a parallelepiped occupying an 18 cm $\times$ 25 cm $\times$ 46 cm volume. The highest water tunnel velocity is 11 cm/s. We take $\rho = 1000$ kg/m³ for the water density and $\eta = 10^{-3}$ Pa s for the dynamic viscosity.

The soft part of the robot, consisting of the flexible fin and tail, was printed in 3D using NinjaFlex filaments from

NinjaTek. This elastic material provides the flexibility of the swimmer's tail, whose shape is shown in Fig. 1(a). Two cables were attached to the tail and guided toward the head through the holes of the flexible skeleton. These two cables were glued to a connecting wheel of the waterproof servomotor HS-5086WP from Hitec, which was fixed to both the 3D printed rigid head and flexible skeleton. This choice obviated the problems of sealing the moving parts of the robot. The rotation of the servomotor releases one cable while pulling the other, thereby mimicking the action of two antagonistic muscles. The back and forth rotation of the servomotor wheel causes both an undulation of the flexible tail and a pitching motion. To simulate an outer skin, we quantified the effect of covering the flexible skeleton with a very thin latex sheet and found that it does not affect the measurements within the precision of our devices. Therefore we kept the skeleton uncovered for simplicity.

The robotic fish is held in place by a rod connected to a force sensor [Fig. 1(b)], a Honigmann RFS®150E, which permits measurement of the total longitudinal force with a precision of approximately 10^{-4} N. The positive and negative values of this force are related to a thrust-prevailing regime and drag-prevailing regime, respectively. In addition, the fish is wired to an Arduino Mega micro-controller, which simultaneously controls the servomotor, acquires data from the force sensor, and drives the fast camera.

We denote U_0 as the free stream velocity, F as the time-averaged value of the longitudinal force, L as the total length of the fish, and S_D as its projected frontal area at rest. Further, the fish's tail consists of a flexible caudal fin with length L_{cf} and height h [Fig. 1(a)]. Unless stated otherwise (two other caudal fins were also tested, cf. Sec. III A), $L = 17$ cm, $L_{cf} = 3.6$ cm, and $h = 6$ cm. The caudal fin has a heterogeneous thickness profile spanning from 1 mm at the contact with the peduncle to 0 mm at its trailing edge. The flow around the fish is characterized by a high Reynolds number, $Re = \rho U_0 L / \eta$, with typical values of $\sim 10^4$.

The head of the fish is fixed by a rod in the reference frame of the lab. The tail of the robotic fish is driven by the deformation of its body through the rotation of the servomotor wheel, whose angular position is denoted as φ . Experiments are performed either with a motionless (φ is constant) or motile ($\varphi(t)$ depends on time) fish.

For the motionless fish, $\varphi = 0^\circ$ corresponds to the fin aligned with the swimmer body. A calibration is then performed without any fluid motion to connect the imposed

FIG. 1. Fish design. (a) 3D representation of the robot. The transparent parts of the robot were printed with a rigid polymer (PLA). The servomotor is drawn in green, and the cables are drawn in red. The soft parts of the robot are opaque and shown in white. The fish has total length L and a tail consisting of a flexible caudal fin with length L_{cf} and height h . (b) Top view of the compliant fish with an imposed tail deformation ($\varphi = 70^\circ$, $A_s = 8.1$ cm), the holding rod, and the three electric wires connecting the servomotor with the micro-controller.

FIG. 2. Image sequence (half of a period) of the swimming fish with the parameters $\Phi = 100^\circ$, $f = 1$ Hz, and $U_0 = 0$.

angular position φ with the deformation of the tail, characterized by the static amplitude of the fin A_s [Fig. 1(b)].

For the motile fish, the swimming dynamics were studied by imposing a harmonic angular function $\varphi(t) = \frac{\Phi}{2} \sin(2\pi ft)$, where Φ is the angular amplitude (range 0° – 140°) and f is the forcing frequency (range 0–1.6 Hz). The tail-beat amplitude A , defined as the tip-to-tip excursion of the tail, is directly measured on the image sequence (Fig. 2). Because of the interaction of the tail with water, the tail-beat amplitude A measured with the motile fish at an angular amplitude Φ might be different from A_s measured with the motionless fish at $\varphi = \Phi/2$. For the motile fish, both the tail-beat amplitude A and average value of the longitudinal force F should be functions of Φ , f , and U_0 .

Images are recorded with a MINICAM WX50 from Photron. A typical sequence centered on the fish trunk and tail is given in Fig. 2. To illustrate the flow in the vicinity of the caudal fin, we implemented a particle image velocimetry (PIV) system along the horizontal medial plane of the fish (see the [supplementary material](#)).

III. RESULTS

We investigate the swimming characteristics of the robotic fish. First, in Sec. III A, we perform experiments in the absence of mean flow ($U_0 = 0$ in the water tunnel) and analyze the amplitude and thrust of the motile fish. Second, in Sec. III B, we exclusively study the drag, i.e., the frictional force exerted on the motionless fish for several static amplitudes and free stream velocities. Finally, in Sec. III C, we characterize the self-propelled swimming conditions defined as the combinations of parameters A , f , and U_0 that result in a zero

average longitudinal force, which corresponds to a thrust–drag balance.

A. Thrust analysis

To investigate the thrust alone, we work with a motile fish and remove the drag by taking the free stream velocity $U_0 = 0$. Figure 3 presents the dependence of the amplitude A with respect to the imposed parameters Φ and f . For a given frequency, the higher the Φ , the higher the A . For a given Φ , the amplitude A seems constant for small f . By increasing the frequency, the amplitude A increases and reaches a maximum before dropping: this is a characteristic of a resonant response. We denote f_m as the resonant frequency for which the amplitude is a maximum. This frequency depends on Φ , and we observe that Φ and f_m are inversely related [Fig. 3(b)].

This result is consistent with what has been seen before with flexible panels,^{11,14–16} but here, we provide the first results for a compliant robotic fish. As in Ref. 15, this inverse relationship appears to be a consequence of the nonlinear behavior of the system.

Measurements of the thrust F_T from the force sensor are shown in Fig. 4(a), where F_T is an increasing function of both Φ and f . Plotting F_T as a function of $A^2 f^2$ in Fig. 4(b) reveals a linear trend over the whole range of values, down to the precision of the force sensor of approximately 10^{-4} N. This scaling holds over 3 orders of magnitudes for the thrust [as shown in the inset of Fig. 4(b)]. Based on Eq. (1), the linear regression leads to $C_T S_T = (134 \pm 2) \times 10^{-4} \text{ m}^2$. This result suggests that our flexible compliant robot exhibits the same $A^2 f^2$ scaling as undulating rigid structures,^{11,14,35,37} irrespective of whether it is resonant or not. This approach conciliates the dynamics of

FIG. 3. (a) A as a function of f for several Φ (20° , 40° , 60° , 80° , 100° , 120° , and 140° from bottom to top). The red arrow is a guide for the eye to display the position of the resonant points. (b) The resonant frequency f_m as a function of Φ .

FIG. 4. Thrust measurements. (a) F_T as a function of f for several Φ . (b) F_T as a function of $A^2 f^2$, with A measured in Fig. 3. The solid line is a linear regression, which leads to $C_T S_T = (134 \pm 2) \times 10^{-4} \text{ m}^2$ in Eq. (1). Inset: log-log plot of the same data.

flexible structures, whose thrust and amplitude are maximal near the same resonant frequency.¹⁶

To validate our observations, we slightly modified the robot so that the caudal fin is interchangeable. Measurements were now performed with three caudal fins ($L_{cf} = 3.6, 6.0$, and 9.2 cm and all with $h = 6 \text{ cm}$) with the same experimental protocol. As seen in the image sequences in Fig. 5(a), the dynamics of fin deformation for the two longest fins are different from the shortest one with $L_{cf} = 3.6 \text{ cm}$. The tip of the shortest fin is always in phase with the motion of the peduncle, while it can be significantly out of phase for the longest fin. For all three fins, the thrust is still proportional to $A^2 f^2$, and the proportionality factor is relatively the same [Fig. 5(b)]. This is consistent with the results obtained by Quinn *et al.* on flexible panels performing a heaving motion.¹⁴ They found that the thrust does not depend significantly on the panel elasticity or geometry nor on the profile of the panel deformation, as long as the deformation is accounted for by the tail-beat amplitude. These observations seem to hold for our compliant robot as well. In this work, we only considered swimmers with a moderate length of the caudal fin, but it would be very engaging to tackle the effects of the deformation modes of long and flexible tails.^{45–47}

B. Drag analysis

To investigate the drag F_D alone without any thrust, measurements were performed with a motionless fish and imposing a flow in the water tunnel. The drag values recorded by the force sensor were negative, and we display the absolute values here for simplicity. To quantify geometrical effects, the tail was placed in various off-center positions, which are characterized by the static amplitude A_s [Fig. 1(b)]. For each value of

A_s , the free stream velocity U_0 is varied from 0 to 0.105 m/s [Fig. 6(a)]. For high Reynolds numbers, F_D is proportional to $U_0^{2.48}$ and defined by

$$F_D = \frac{1}{2} \rho C_D(A_s) S_D U_0^2, \quad (2)$$

where S_D is a reference area for the drag and $C_D(A_s)$ is the drag coefficient. For a measured F_D and an imposed A_s , $C_D(A_s) S_D$ is computed using Eq. (2). For the centered tail, the measurements lead to $C_D(0) S_D = (25.2 \pm 0.3) \times 10^{-4} \text{ m}^2$. By using the wetting area as a reference for S_D (approximated to 135 cm^2), we estimate $C_D(0) = 0.19$. We then plot $C_D(A_s)/C_D(0)$ as a function of A_s in Fig. 6(b) and observe a parabolic trend. An off-centered tail augments the projected frontal area of the fish, resulting in a drag magnification when A_s becomes large enough. But this relative increase in the projected frontal area, which is proportional to A_s , is not enough to explain the quadratic behavior of the drag coefficient seen in Fig. 6(b). An analogy can be made with airfoils, whose drag coefficient is a quadratic function of the incidence angle.⁴⁹ The trend observed in Fig. 6(b) is captured by the simple parabola $1 + (A_s/A_0)^2$, where A_0 is the length. Although this correction might be understood as ad hoc, we rationalize this quadratic behavior by considering the symmetry $A_s \rightarrow -A_s$ in the Taylor expansion of the drag coefficient within the small A_s limit. This development permits to account for both the effect of the projected frontal area and of the geometry in the determination of $C_D(A_s)$.

The best fit gives $A_0 = 0.089 \pm 0.002 \text{ m}$. Therefore, we can infer the following expression for the drag force:

$$F_D = \frac{1}{2} \rho C_D(0) S_D U_0^2 [1 + (A_s/A_0)^2]. \quad (3)$$

FIG. 5. Effect of the caudal fin design on the thrust. (a) Image sequence over one period for each fin with the parameters $\Phi = 60^\circ$, $f = 1.2 \text{ Hz}$, and $U_0 = 0$. (b) F_T as a function of $A^2 f^2$.

FIG. 6. Drag force measurements. (a) F_D as a function of U_0^2 for several static fin positions (A_s varies between 0 and 0.081 m). Solid lines are linear fits for the two extreme values of A_s . Inset: log-log plot of the same data. (b) $C_D(A_s)/C_D(0)$ as a function of A_s . A parabolic fit $C_D(A_s)/C_D(0) = 1 + (A_s/A_0)^2$ leads to $A_0 = 0.089 \pm 0.002$ m.

The drag exerted on a static fish exhibits nonlinear behavior connected to the amplitude A_s of the caudal fin. The length A_0 characterizes this effect, and it is not surprising that its value has the same order of magnitude as the length of the fish. The drag expression for a motionless fish will be useful to compare with the dynamic case of a motile fish below.

C. Self-propelled swimming conditions

Experiments are now carried out with a motile fish and an imposed flow in the water tunnel. The average force F and the tail-beat amplitude A are functions of Φ , f , and U_0 . For a given velocity U_0 , the force is represented as a 2D map displaying both the thrust-prevailing (positive values) and the drag-prevailing (negative values) zones [Fig. 7(a)].

At the frontier of these two regions, the thrust and drag counterbalance each other ($F = 0$), and the fish is self-propelled. Each set of parameters (A , f , U_0) fulfilling $F = 0$ defines a self-propelled condition. All of them are represented in Fig. 7(b): for a given U_0 , larger f results in smaller A , while increasing U_0 shifts the points of self-propulsion toward the larger values of A and f .

To account for these measurements, we use the expressions of the thrust and drag obtained independently [Eqs. (1) and (3)] and assume that both expressions still hold for the self-propelled swimming conditions. For the drag expression, we replace A by A_s . The balance $F_T = F_D$ leads to

$$\frac{Af}{U_0} = St = St_0 \sqrt{1 + \left(\frac{A}{A_0}\right)^2}, \quad (4)$$

with $St_0 = \sqrt{\frac{C_D(0)S_D}{C_T S_T}}$ as the Strouhal number at a small amplitude ($A \ll A_0$). This expression is tested with the self-propelled data by plotting the Strouhal number $St = Af/U_0$ as a function of A (Fig. 8). All data collapse on the same curve regardless of the free stream velocity or tail-beat frequency. Consistent with Eq. (4), the Strouhal number is an increasing function of the tail-beat amplitude only.

A quantitative comparison with the model obtained from the independent thrust and drag analyses ($St_0 = 0.43 \pm 0.01$ and $A_0 = 0.089 \pm 0.002$ m, black dashed line in Fig. 8) shows that the self-propelled data systematically lead to higher values of the Strouhal number. Actually a two-parameter interpolation with these data instead yields $St_0 = 0.52 \pm 0.05$ and $A_0 = 0.052 \pm 0.005$ m (red solid line in Fig. 8). There is a 19% difference between the values of St_0 . Given the lack of data for $A < 0.02$ m and in the limit $A \ll A_0$ and since the error bars of the data points are greater for smaller A , it is not clear whether or not this difference is significant nor whether the extrapolation toward $A = 0$ would lead to a smaller difference or not. In this context, we prefer to say that our model without free parameters validates the independent thrust and drag analyses at small amplitudes within a margin of 19%.

Unlike the small amplitude regime ($A \ll A_0$), the intermediate amplitude regime ($A \sim A_0$) is characterized by numerous

FIG. 7. (a) Time-averaged force F as a function of Φ and f for a given free stream velocity $U_0 = 0.042$ m s⁻¹. The red line corresponds to the pairs of (Φ, f) values for which F equals zero. (b) Self-propelled conditions for several values of U_0 .

FIG. 8. Strouhal number $St = Af/U_0$ as a function of A for the data of Fig. 7(b). The red solid line is a two-parameter interpolation with the full model, $St_0\sqrt{1 + (A/A_0)^2}$, and the black dashed line with the gray error bars is the model with the parameters inferred from the independent thrust and drag analyses.

data points, and all of them have significantly larger values than the prediction based on the independent thrust–drag analyses. For the largest values, the difference reaches almost 50%. This means that the approach does not hold any more and that the motile fish gives rise to a larger effective drag than expected from the independent thrust–drag analyses. Actually the difference between the static drag and the real drag experienced by the motile fish is characterized by almost a factor two between the two predictions of A_0 . This difference might be partly understood by considering the effective frontal area as one possible contribution: for instance, in the limit of high frequencies, one could model the swimmer as a solid object, whose increase in effective frontal area with the fin amplitude is doubled compared to the static case as the fin is flapping on both sides of the fish.

IV. DISCUSSION AND CONCLUSIONS

Our analysis shows that the Strouhal number of a self-propelled flexible swimmer is not necessarily constant but an increasing function of the tail-beat amplitude. At small amplitudes, the model based on the independent analyses of thrust and drag is rather good, given that there is no free parameter. These results provide evidence that the Strouhal number of a self-propelled swimmer is associated with the thrust–drag balance^{22,27} and that performing independent analyses is relevant to characterize the swimming properties to the leading order.

For biological swimmers, the Strouhal number appears to be a constant over a large number of decades of the Reynolds number,²² as well as the second dimensionless number, $A/L \sim 0.2$.^{26–29} This constancy might be rationalized with a principle of minimum energy consumption.²⁷ Unfortunately it was not possible to measure the energy consumption of our system. Nevertheless, our experimental setup has permitted to explore the effect of A/L , and we demonstrate that the Strouhal number is amended by a correction factor induced by the amplitude on the effective drag.

The values of our Strouhal number at small amplitudes are found around 0.5 in comparison with natural swimmers (average on the order of 0.3^{22,27}). This result suggests that

the thrust coefficient is smaller and/or the drag coefficient is larger than those of an average natural swimmer. The fact that the motion of the robot is a pure pitch could explain this observation. Adding a degree of heaving in the design of the robot might decrease the values of the Strouhal number.

SUPPLEMENTARY MATERIAL

See [supplementary material](#) for movies illustrating the tail oscillation of the motile fish (3 caudal fins, $L_{cf} = 3.6, 6.0$, and 9.2 cm, with the parameters $\Phi = 60^\circ$, $f = 1.2$ Hz, and $U_0 = 0$) and the flow in the vicinity of the caudal fin by particle image velocimetry.

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