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Numerical simulation of impact and compression after impact of asymmetrically tapered laminated CFRP



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A B S T R A C T

This paper presents a numerical simulation of impact and compression after impact (CAI) of a tapered composite laminate using a discrete ply model. Three types of damage: matrix cracking, delamination and fiber rupture are considered in the model. The presence of ply drop-off generates some discontinuities in the stress field and therefore adds difficulties to the simulation.

Analyses of numerical results are performed to understand the damage and failure mechanisms in both tests. Numerical results in terms of force-displacement curves, delamination shape, CAI displacement field and residual strength are compared with experimental data. Impact simulation is in good correlation with the tests. CAI strength is under predicted and depends on quality of the meshing of the transition region. This study highlights the importance of modeling intra-ply matrix cracking for impact simulation.

Keywords:

Composite

Ply drop-off

Impact

Compression after impact

Finite element simulation

1. Introduction

Impact damage tolerance of CFRP materials remains an important concern for structure design. Impact damage which is mainly characterized by matrix cracking, delamination and fiber rupture, usually propagates far beyond the impact point. Such damage is mostly present inside the laminate and is visually hard to detect from outside [1,2]. Even in the case of low velocity/low energy impact, the residual compression strength can be significantly affected. For thin composite panels, the energy threshold for visual detection of external damage (permanent indentation) corresponds already to an important loss of compression strength [3,4]. Since some aeronautical structures are inclined to tool drop and small debris impacts during their life cycle, the requirements and regulations by aviation authorities include a concept of impact damage tolerance.

Thickness tapering is commonly used to reduce the weight of composite structures, though it leads to the creation of resin rich pocket at the end of a dropped-off ply. Due to such material discontinuities and the local curvature of continuous plies, the resulting structure may fail prematurely under static or fatigue loadings

[5–9]. Damage, generally in terms of matrix cracking and delamination, is triggered by a high stress concentration around the ply drop-off.

In the literature, many authors have developed numerical models for impact of composite laminates [10–14]. Depending on their level of complexity, the models are capable of reproducing some or all major types of damage: matrix cracking, delamination and fiber rupture. The meso-scale level initially introduced by Ladevèze and Allix [15] enables a faithful reproduction of impact type damage. Regarding the material laws, failure damage mechanics is generally used to simulate intra-laminar damage [11,16]. The use of cohesive elements with energy based damage model is now being used in many works to model discrete failure like delamination [17,18]. Some authors [12,19] also use it to model discrete matrix cracking and therefore define an indirect coupling between matrix cracking and delamination.

Regarding the prediction of residual strength, both analytical [20–22] and FE models [23–25] have been proposed. In some papers [23], initial impact damage is artificially introduced in the model prior to the simulation of compression. Others [24,25] are more faithful using a two-step model: an impact damage is simulated first and then followed by a simulation of the compression test.

To our knowledge, neither impact nor CAI simulation of tapered composite has been reported in the literature. This study pursues the work of Bouvet et al. [12,14,24] to develop a robust numerical model DPM (Discrete Ply Model) for predicting impact damage tolerance; i.e. simulation of impact and CAI. It is also a continuation

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of the impact of tapered laminates study described in [26] where an analysis of low velocity impact response of tapered laminates is presented. The study highlights a strong similarity with plain laminates in terms of damage mechanism. In this paper, the DPM is used to simulate the impact and CAI of a tapered specimen. The results of the numerical simulation are compared with experimental data for model validation and analysis of damage mechanisms.

2. Numerical modeling

The modeling approach is based on the work of Bouvet et al. [27]. They developed a discrete 3D FE model at the meso-scale for the simulation of impact damage of composite laminate. It has then been pursued to simulate the CAI test [24]. In this study, the capability of the model is extended to simulate both impact and CAI of a laminate with multiple internal ply drop-offs. To do so, a meshing process of the ply drop-off region has been developed and some changes regarding the material laws have been done compared to [14].

2.1. Meshing

The laminate meshing is presented in Fig. 1. It is meshed at the ply level with C3D8 volume elements. Each element represents, in the thickness direction, two plies with the same orientation. The ply elements are connected with zero thickness cohesive elements to model both delamination and transverse matrix cracking. To ease inter-ply connection, the nodes network needs to be uniform throughout the laminate which causes a twist of 45° and -45° ply elements (Fig. 1b). The element size for 0° and 90° plies is 1.25 × 1.25 mm² in the laminate plan.

One of the main challenges in impact modeling is to account for complex interactions between different types of damage. In the case of the DPM, the coupling between transverse matrix cracking and delamination is modeled through the laminate meshing. It is one of the main building blocks of the model. Cohesive elements for

matrix cracking are parallel to the ply fiber orientation and defined in between each volume element. As a result, volume elements are disconnected along the ply transverse direction in case of matrix cracking. Such geometric discontinuity is used to capture indirectly the coupling between matrix cracking and delamination.

Modeling of the drop-off ply region is illustrated in Fig. 2a. The meshing represents the sub-laminate of the ply drop-off area shown in Fig. 2b. In this example, ply number 2 is terminated and plies 1 and 3 are continuous. In the thick section, inter-lamina cohesive elements are terminated at the end of the dropped-off ply. Then, a new group of cohesive elements are used to connect the continuous plies in the thin section. Note that there is no volume or cohesive elements in the resin pocket. This hypothesis is similar to assuming this area is already damaged before impact. In reality, it must be highly pre-stressed due to curing residual stress. So, to avoid any further complication of the model or longer computing time, it seemed not reasonable to develop a detailed meshing strategy of the area. In Reference [28], similar assumption has been used to simulate a tapered laminate under in-plane load. In the case of impact, the assumption is more valid since the emphasis is on the propagation of delamination rather than its initiation.

2.2. Material laws

The impact is simulated with the Abaqus® 6.11 explicit/dynamic solver and all material laws are defined in a user-subroutine VUMAT. This part describes the non-linear modeling of the different damage types: fiber failure, matrix cracking and delamination.

2.2.1. Fiber failure

Modeling of fiber failure for impact simulation requires particular considerations. In fact, fiber may fail either under traction or compression load. In both cases, a high amount of energy is released as shown in the material property of Table 1. This fiber failure energy release rate (ERR) needs to be taken into consideration to

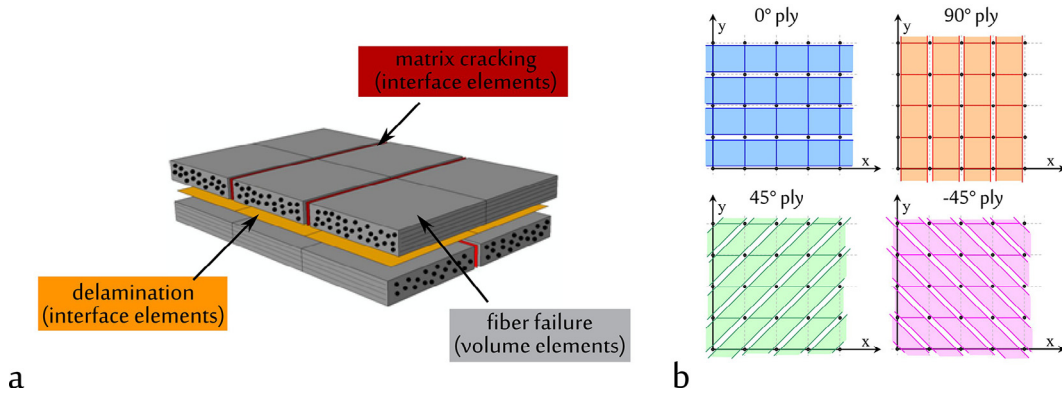


Fig. 1. Description of the Discrete Ply Model: (a) different types of damage that can be represented and (b) meshing strategy.

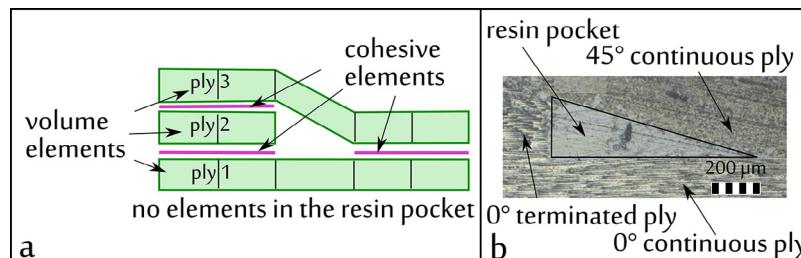


Fig. 2. (a) meshing around the resin pocket, (b) micrograph of a ply drop-off.

Table 1
Mechanical properties of T700GC/M21 [29,30].

ρ	Density	1600 kg/m ³
E_f^t	Tensile Young's modulus in fiber direction	130 GPa
E_f^c	Compressive Young's modulus in fiber direction	100 GPa
E_t	Transverse Young's modulus	7.7 GPa
G_{lt}	Shear modulus	4.8 GPa
ν_{lt}	Poisson's ratio	0.33
$\varepsilon_f^{t,0}$	Tensile failure strain in fiber direction	0.016
$\varepsilon_f^{c,0}$	Compressive failure strain in fiber direction	-0.0125
σ_f^{crush}	Mean crushing stress	270 MPa ^a
σ_t^{rupt}	Transverse tensile strength	60 MPa
τ_{lt}^{rupt}	In-plane shear strength	110 MPa
G_I^c	Mode I delamination critical ERR	0.6 N/mm
G_{II}^c and G_{III}^c	Mode II and III delamination critical ERR	1.5 N/mm
$G_{I,f}^{c,t}$	Fiber traction critical ERR	133 N/mm ^b
$G_{I,f}^{c,c}$	Fiber compression critical ERR	40 N/mm ^c

^a [31],

^b Material T300/913 [32],

^c Predicted value from [14].

compute the energy dissipation in the model. In reality, fiber failure is a discrete phenomenon and it should therefore be computed using failure damage mechanics within an interface element. Though, such option would significantly increase the number of degrees of freedom in the model. For this reason, it is preferred to manage fiber failure ERR inside volume elements using mesh-size independent formulation. Such formulation has been initially developed by Bazant and Oh [33] within the crack band theory.

In the model, the critical ERR is proportionally dissipated through the 8 integration points of the volume element based on the formulation of Bouvet et al. [14,34]. During the impact simulation, it is possible to encounter a single ply bending after delamination of its interfaces. In other words, some of the integration points are subjected to tension while the others are in compression. Therefore, the element damage evolution under traction and compression failure should not be formulated separately and a coupling between both failure modes needs to be considered. The energy dissipated in such type of loading is very complicated to measure experimentally, so, due to the lack information, a linear contribution of the critical ERR (Eq. (1)) is used in the model.

$$\frac{G_{I,I}^t}{G_{I,I}^{c,t}} + \frac{G_{I,I}^c}{G_{I,I}^{c,c}} \leq 1 \quad \text{Eq. 1}$$

where $G_{I,I}^{c,t}$ ($G_{I,I}^{c,c}$) is the critical mode I ERR in longitudinal direction in tension (compression) and $G_{I,I}^t$ ($G_{I,I}^c$) is the mode I ERR in longitudinal direction in tension (compression), defined as:

$$G_{I,I}^t = \frac{1}{S} \int_V \int_0^{\varepsilon_f^{t,1}} H_{II}^t (1-d) \langle \varepsilon_I \rangle^+ d\varepsilon_I \cdot dV \quad \text{Eq. 2}$$

$$G_{I,I}^c = \frac{1}{S} \int_V \int_0^{\varepsilon_f^{c,1}} H_{II}^c (\varepsilon_I - \varepsilon_I^p) d\varepsilon_I \cdot dV$$

where ε_I (σ_I) is the longitudinal strain (stress) in longitudinal direction, $\langle x \rangle^+$ ($\langle x \rangle^-$) the positive (negative) value of x , V the element volume, S the element section normal to longitudinal direction and $\varepsilon_f^{t,1}$ ($\varepsilon_f^{c,1}$) the longitudinal strain at element failure in traction (compression), d the damage variable and H is the homogenized stiffness matrix of the ply with H_{II}^t (H_{II}^c) the traction (compression) value in longitudinal direction.

The degradation of the longitudinal stiffness of the element is governed by a single variable d , defined in such a way that the inequality in Eq. (1) is verified at element failure. Since some integration points of an element may be in traction and the others in compression and the failure strain in traction of the material is different from the compressive one, it is more convenient to define

the damage variable d in terms of relative strain. Considering the relative strain e defined as $\frac{\varepsilon_I}{\varepsilon_f^{t,0}}$ in traction or $\frac{\varepsilon_I}{\varepsilon_f^{c,0}}$ in compression,

where $\varepsilon_f^{t,0}$, $\varepsilon_f^{c,0}$ are respectively the material longitudinal failure strains in traction and in compression. Then d is defined in Eq. (2) to obtain a linear evolution of the element longitudinal stress until failure:

$$d = \frac{e_1(e_{max} - e_0)}{e_{max}(e_1 - e_0)} \quad \text{Eq. 3}$$

where

$$e_{max} = \max_{i=1}^8 (e^i) \quad \text{Eq. 4}$$

With e^i the element dimensionless strain at integration point, e_0 the element maximum relative strain at integration point at damage initiation and e_1 the element maximum relative strain at element failure (when $d = 1$) computed by resolving Eq. (1). It can be noticed that e_0 can be smaller than 1 because the instant of failure initiation is computed from strain at the element nodes to better account for the bending of the ply. In traction, the longitudinal stress is computed as:

$$\text{If } \varepsilon_I > 0, \quad \sigma_I = H_{II}^t(1-d)\varepsilon_I + H_{It}(1-d)\varepsilon_{It} + H_{Iz}(1-d)\varepsilon_{Iz} \quad \text{Eq. 5}$$

where H_{II}^t , H_{It} , H_{Iz} are the matrix stiffness components. It can be noticed that H_{II} is not the same in tension (H_{II}^t) and in compression (H_{II}^c) due to asymmetric elastic behavior of composite (Table 1).

In compression, the calculation of the stress is more complex because after failure, fiber crushing needs to be modeled with a plastic law. If the compressive stress is evaluated similarly to Eq. (4), there would be a discontinuity of the longitudinal plastic strain when the law changes from damage mechanics to plasticity. To overcome this problem, an equivalent plastic strain is computed during the damage evolution so that the element still satisfies the equality of Eq. (1) at failure. By identification, the equivalent plastic strain during damage evolution is:

$$\text{If } \varepsilon_I < 0, \quad \varepsilon_I^p = d\varepsilon_I \quad \text{Eq. 6}$$

Afterwards, the compressive stress is computed as:

$$\text{If } \varepsilon_I < 0, \quad \sigma_I = H_{II}^c(\varepsilon_I - \varepsilon_I^p) + H_{It}(1-d)\varepsilon_{It} + H_{Iz}(1-d)\varepsilon_{Iz} \quad \text{Eq. 7}$$

During crushing, the plastic strain is determined using a yield function f (Eq. (8)) and the material mean crushing stress σ_f^{crush} . This property has been measured by Israr et al. [31] from in-situ experimental tests:

$$\varepsilon_I^p \text{ so that } f = |\sigma_I - \sigma_f^{crush}| \leq 0 \quad \text{Eq. 8}$$

Fig. 3 summarizes the behavior of volume elements in the fiber direction. The example shows a case where some part of the energy is dissipated in traction and some in compression.

2.2.2. Matrix cracking

Matrix cracking is represented by the intra-lamina cohesive elements. The location is therefore imposed by the element mesh-size. Note that the model does not aim to reproduce the dense network of crack generally observed inside impacted laminate. As a result, it cannot estimate the energy dissipated through matrix cracking. The goal is only to be able to represent the discontinuity generated by larger cracks and at the same time to trigger or pilot delamination. The criterion for matrix cracking is computed inside volume elements. It is based on Hashin's failure criterion:

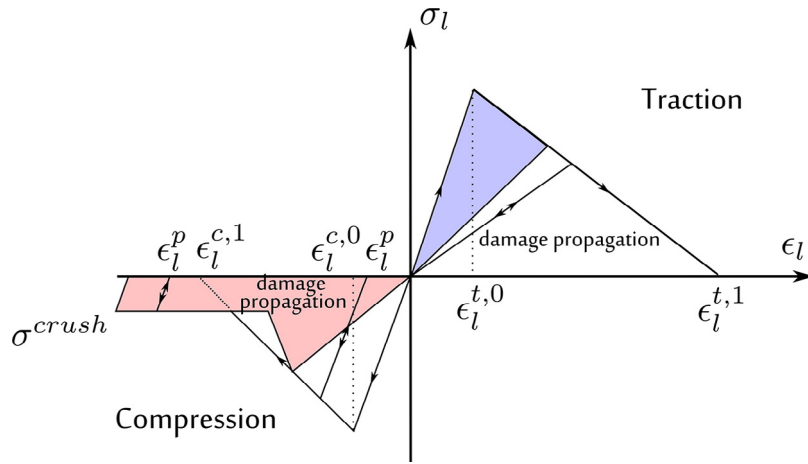


Fig. 3. Volume element behavior in the fiber direction.

$$\left(\frac{\sigma_t^+}{\sigma_t^{rupt}}\right)^2 + \frac{(\tau_{lt})^2 + (\tau_{tz})^2}{(\tau_{lt}^{rupt})^2} \leq 1 \quad \text{Eq. 9}$$

where σ_t , τ_{lt} , τ_{tz} are respectively the transverse, lt and tz shear stresses, and σ_t^{rupt} (τ_{lt}^{rupt}) the transverse (shear) failure stress. When the criterion is met in a volume element, the shear and traction strength of the neighbor cohesive elements are vanished. The compressive strength is however maintained to prevent volume elements from inter-penetrating. Moreover, the permanent indentation is implemented in the matrix cracking element. The formulation is based on matrix crack non-closure due the containment of debris. The reader is suggested to refer to References [34,35] for a detailed description and validation of the law.

2.2.3. Delamination

Modeling of delamination is achieved by reducing the stiffness of the inter-lamina cohesive elements. Element damage is initiated by a quadratic stress criterion taking into account modes I, II and III of propagation. The crack opening is governed by fracture mechanics. A linear contribution of the three modes is used considering the same value of ERR for modes II and III.

$$\frac{G_I}{G_I^c} + \frac{G_{II}}{G_{II}^c} + \frac{G_{III}}{G_{III}^c} = 1 \quad \text{Eq. 10}$$

3. Experimental setup

3.1. Material and specimens

Two tapered specimens made of T700GC/M21 carbon-epoxy are considered. The material is available as a unidirectional prepreg tape of 0.25 mm thick. Material mechanical properties are presented in

Table 1. Specimens are cured in autoclave following the manufacturer's recommendations.

An asymmetrically tapered specimen is considered; i.e. only one side is tapered and the opposite one remains flat (Table 2). Twenty four plies constitute the layup of the thick section with a stacking sequence of $[45_2, 0_2, -45_2, 0_2, 90_2, 0_2]_s$. Then, 4 groups of plies, all oriented in 0° are dropped-off in the mid-plane of the specimen resulting in a thin section stacking of $[45_2, -45_2, 0_2, 90_2]_s$. The thickness changes from 6 mm to 4 mm. The plies are grouped in pairs to reduce the number of interfaces and the size of the numerical model. Plies that belong to the same group are simultaneously dropped off. The groups of plies are terminated in an interspersed configuration as shown in Table 2. The step spacing between two consecutive drop-offs is 5 mm.

The length of the specimens is 100 mm longer than conventional low velocity impact specimen ($100 \times 10 \text{ mm}^2$) [36] in order to allocate spaces for tabs at each of its end. The use of tabs is necessary during CAI test to compensate for the thickness difference between the thick and the thin section (Figs. 4 and 5). The transition section is situated at 125 mm from the edge of the thin section (Fig. 4).

3.2. Tests procedure

Impact tests are performed on a drop weight testing rig. The impactor weights 4 kg and has an hemispherical shape indenter of 16 mm-diameter. Two specimens are impacted at 30 J-energy. The evolution of the impact force is measured by a piezoelectric sensor embedded in the impactor and the impact initial velocity is obtained from an optical sensor. Impact data acquisition is performed at a frequency of 250 kHz. The data are then used to compute the force-displacement graph of the test.

Table 2
Specimen layup.

Layup	[45 ₂	0 ₂	-45 ₂	0 ₂	90 ₂	0 ₂] _s	
Termination order	-	2 nd	-	-	-	1 st	

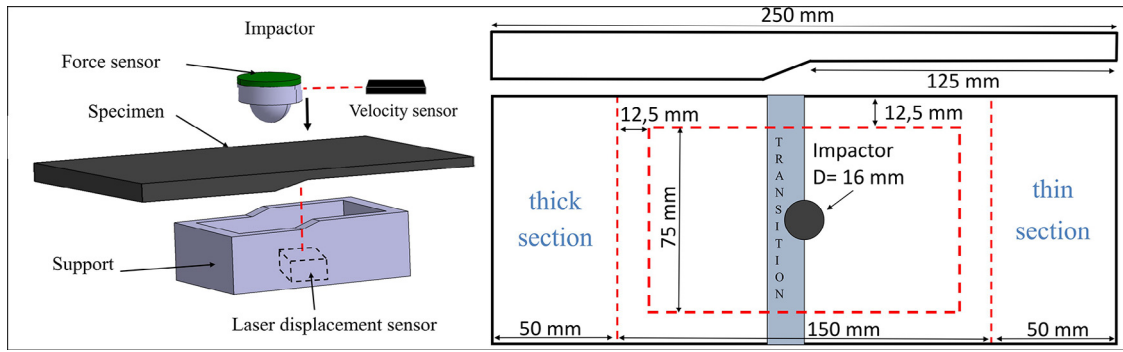


Fig. 4. Schematic of impact boundary conditions.

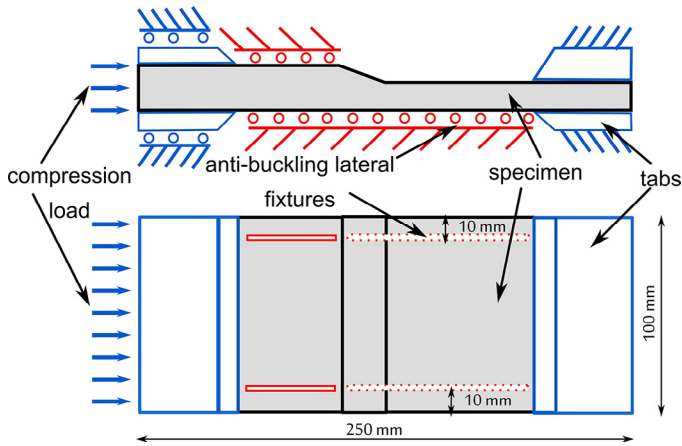


Fig. 5. Schematic of CAI boundary conditions.

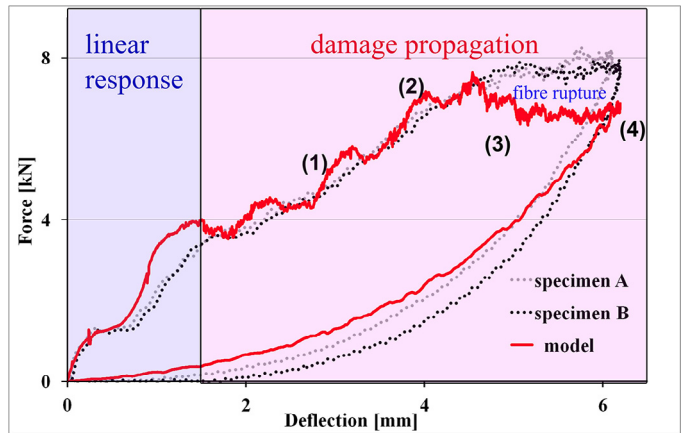


Fig. 6. Comparison of force-displacement graphs.

The impact is realized on the flat side of the specimens. The tapered side is simply supported by a tapered rectangular opening of $125 \times 75 \text{ mm}^2$. This boundary condition is very similar to the Airbus standard AITM 1-0010 [36] defined for plain specimens. The schematic of the impact test setup is illustrated in Fig. 4.

After the impact test, the impacted face is painted in white with black speckle pattern to measure permanent indentation with an image correlation system. The speckle pattern is also used to monitor deformation of the specimens during the CAI test. Then delamination is measured using C-scan technique from the impacted side of the specimen.

Regarding the CAI boundary conditions (Fig. 5), longitudinal supports are provided on both sides at 10 mm from the edges to prevent premature failure due to extensive bending. On the tapered side, the supports are only present in the thick section. Note that the gap of the specimen midline on both sides of the transition region generates a bending couple when it is in-plane loaded. If the specimen were bearing loaded like in the AITM 1-0010 standard or the international standard ASTM D7137/7137M [37], it would result in a premature failure due to the absence of rotational constraints at the boundaries.

4. Results comparison and analysis

4.1. Impact force-displacement graphs

The comparison of force-displacement curves (Fig. 6) gives an overall quality of the impact simulation. The similarity of the curves shows that the model is able to reproduce the global response of

a tapered specimen. It is interesting to notice that the modeling choice of the resin pocket does not affect much the result.

Three distinct phases can be identified from the experimental curves. In the beginning, the plate exhibits an elastic dynamic response until the contact force reaches around 3.75 kN. Then, the loss of bending stiffness implies an onset of important damage which is generally associated with large matrix cracking and delamination. When the contact force reaches about 7.8 kN, an even larger loss of bending stiffness is observed due to fiber failure. Since all three types of damage are observed, the tests represent a good candidate to assess the capabilities of the model.

Regarding the numerical graph, the model predicts well the onset of delamination. Then, the evolution of the force during the second phase is also well reproduced. However, the drop of the contact force at 4.6 mm deflection is more important in the simulation, meaning that fiber rupture is over-predicted by the model. As a result, the numerical graph is 1 kN below experimental graphs in the final plateau. It can be due to a misvaluation of the mean crushing stress or the critical ERR in traction or compression fiber rupture. Increasing one of these parameters will improve the contact force after fiber rupture. Note that measuring the critical ERR is very complicated experimentally. The value of traction rupture critical ERR is obtained from T300/913 material [32] and the compression one was set to match the impact results on 7 different layouts in [14].

4.2. Impact delamination

In Fig. 7, interfaces delamination (Fig. 7 (a) and (b)) obtained from C-scan, are compared with the simulation (Fig. 7 (d)). Numerical

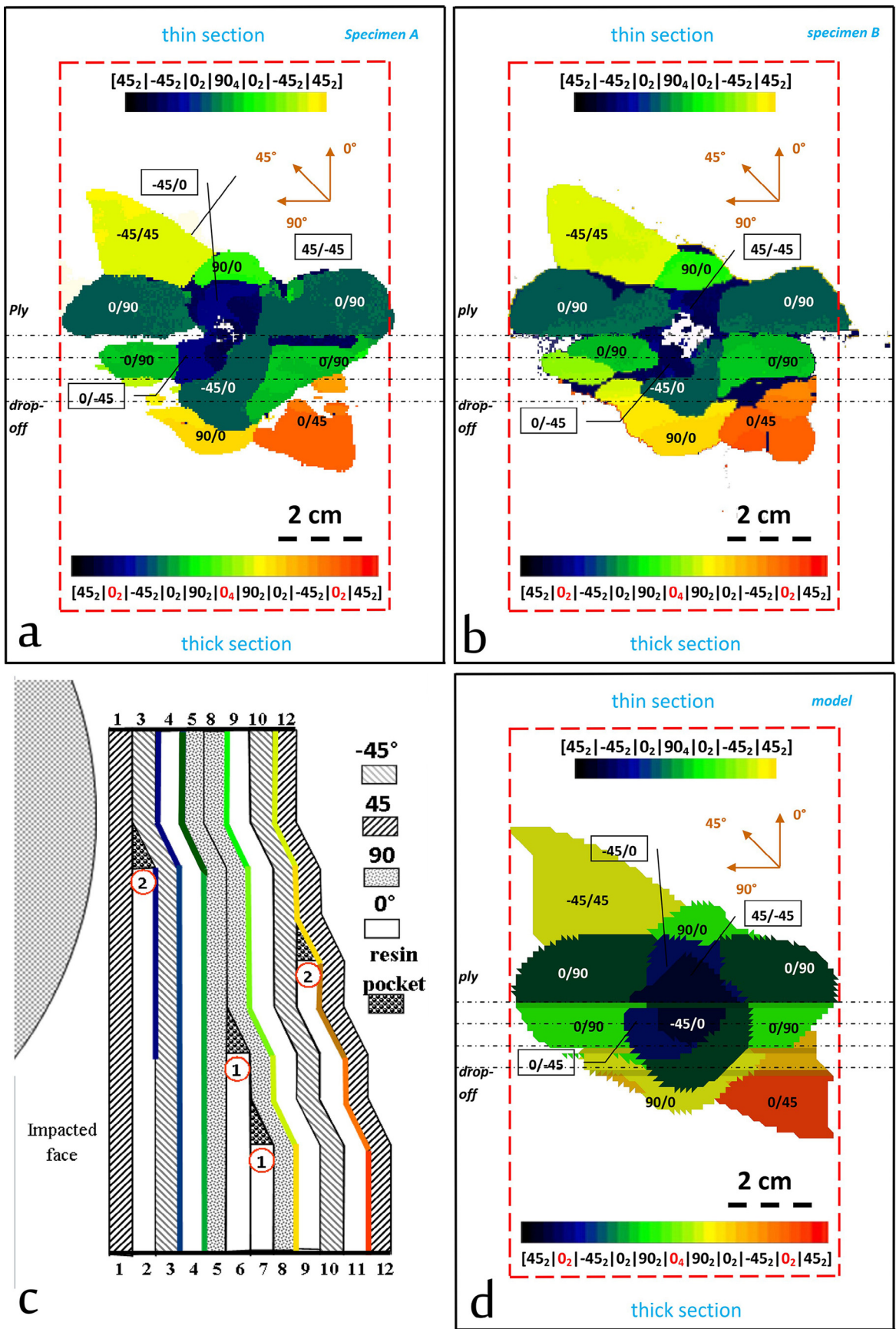


Fig. 7. Comparison of C-scan of delamination: (a, b) Experimentally obtained delaminations for two different experiments, (c) delaminations profile and (d) numerically obtained delaminations.

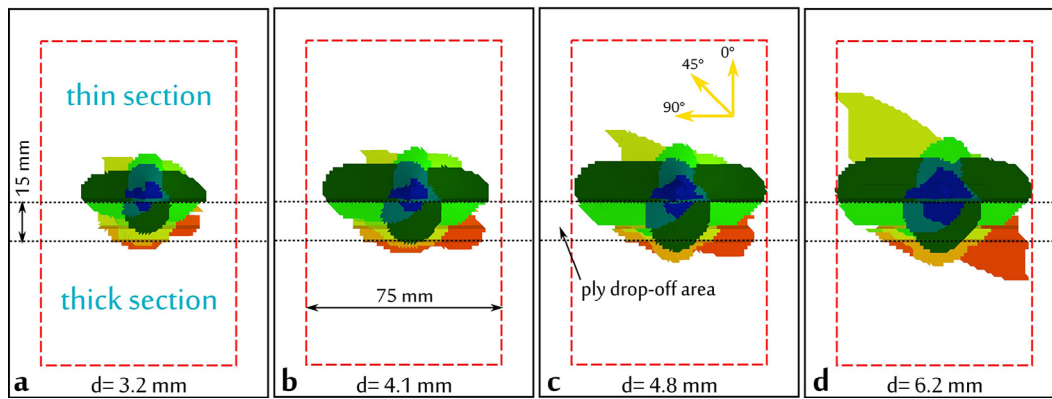


Fig. 8. Propagation of delamination during impact.

results are processed with the same color spectrum to obtain an equivalent C-scan output. Experimental results show low scattering of delamination. Note that the thickness variation on either side of the transition region can change the color of a given interface. For example, the color of interface 0/90 changes from dark green to green. The delamination profile, provided in Fig. 7 (c), helps in the analysis of the results.

With regards to delamination propagation, regardless of the presence of plies drop-off, the direction of propagation always follows the orientation of the underlying ply. Also, the interface of continuous plies is more critical like the 0/90 or the -45/0. The morphology of the delamination predicted by the model is in good agreement with the experiment. Area of projected delamination is respectively equal to 2650 mm² and 2920 mm² for specimen A (Fig. 7a), B (Fig. 7b) and the simulation gives 3000 mm² (Fig. 7d). Therefore, the model is robust enough to provide a good prediction of the delamination shape in spite of the perturbation created by the transition region.

Experimental test provides only limited information about damage mechanism, thanks to the simulation, additional information can be retrieved. For example, in Fig. 8, the progression of delamination throughout the simulation is presented. The images are taken at 4 distinct states of the impact. Fig. 8 (a) and (b) are taken prior to the major fiber rupture at respectively 3.2 and 4.1 mm of deflection. At first, delamination propagates toward all direction at the same rate. Then, the propagation toward the thick section is slowed down once it reaches the end of the ply drop-offs area. At the same time, the propagation in the transverse direction is emphasized. In Fig. 8 (c) and (d), the onset of major fiber rupture stops the propagation in the transverse direction and triggers the propagation of the 0/45 and -45/45 out-most interfaces of the specimen.

4.3. Impact energy dissipation

Contact force and energy dissipation history from the simulation are plotted in Fig. 9. This figure gives an insight view of the damage mechanism during impact. Total damage dissipation graph corresponds to the sum of energy dissipated in delamination and fiber rupture. It equals 17 J which is about 46% of the impact energy. This value is lower compared to the 20 and 21 J dissipated in the tests, obtained from the area below the force-displacement curve. It is clearly observed in the comparison of force-displacement curves in Fig. 6, the area below the curve is smaller for the model. In the model, the post-impact kinetic energy of the plate is negligible (less than 0.1 J) and about 12.6 J are retransmitted to the impactor. Although the model does not account for the dissipated energy in matrix cracking, this contribution alone cannot compensate the difference of 3 to 4 J between the tests and the model. The difference

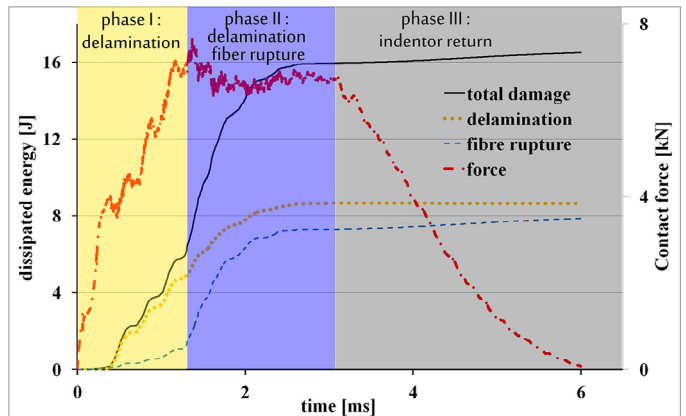


Fig. 9. History of the energy dissipation during impact.

can be due to critical ERR in fiber rupture which is too low or the modeling of permanent indentation which affects the unloading.

In the simulation, approximately half of the dissipated energy is in delamination and the other half is in fiber rupture. It can also be confirmed that most of the energy dissipated in the first phase is due to delamination. Only a small amount is dissipated in the rupture of fiber localized below the indenter. In the second phase, the progression of delamination continues at the same rate and it is accompanied with large fiber rupture. Moreover, the drop of contact force coincides with the increase of energy dissipation in fiber rupture.

Fig. 10 shows the contribution of each mode of propagation in the delamination energy. In the center of the specimen, it is predominately in mode II and III while it is in mixed mode near to the specimen edges with a 50 % contribution of mode I. These data corroborate the need to model both shearing and opening propagation modes during impact.

4.4. CAI force-displacement curves

Simulation of CAI test is realized with the same model successively to impact; some vibration and rigid body motion of the specimen needs to be reduced before application of the compression load, though. To do so, multiple cycles of freeze-release of all degrees of freedom are applied. Concretely, in one cycle, all degrees of freedom are locked during 10⁻⁶ s and then released during 10⁻⁴ s. Then, the compression test is simulated with a controlled displacement speed of 0.2 m/s. A relatively high loading speed needs to be used because of the explicit method; however, we checked that it

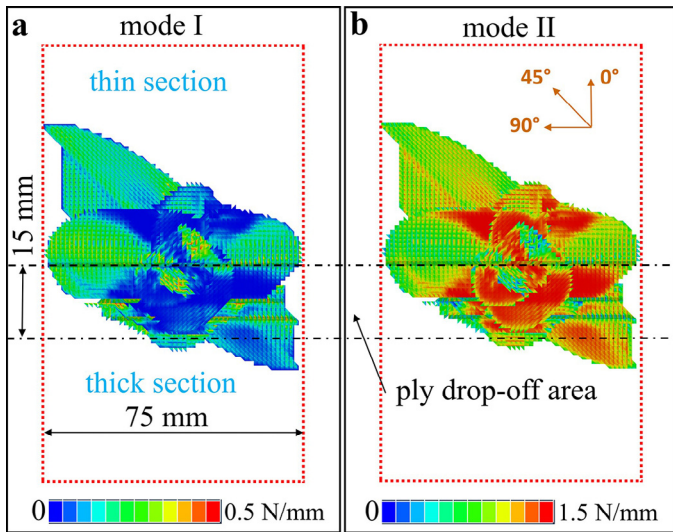


Fig. 10. Delamination propagation in modes (a) I and (b) II and III.

does not induce any significant dynamic effects on the simulation results.

Fig. 11 compares experimental and numerical force-displacement curves in CAI of two different simulations: “model_v1” is the reference model, for “model_v2”, a smoother mesh of the transition region (Fig. 12b) is used but it still does not match the real shape (Fig. 12c). For both experimental tests, the response is linear until 40 kN and failure occurs at around 69 kN. Experimental and numerical curves are quite similar in the beginning though failure load

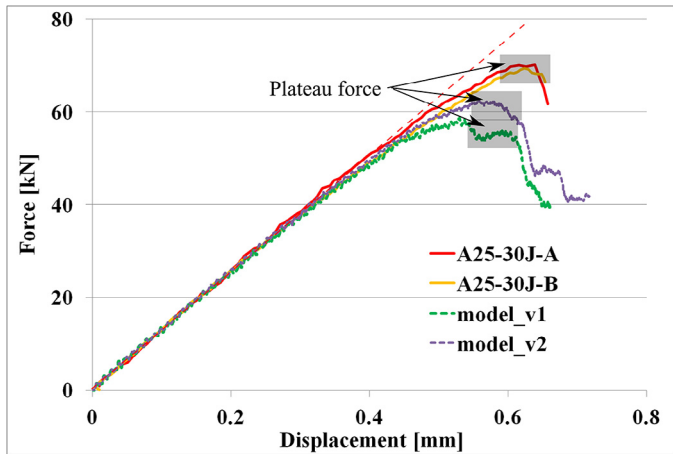


Fig. 11. Comparison of CAI force-displacement curves between tests and simulation.

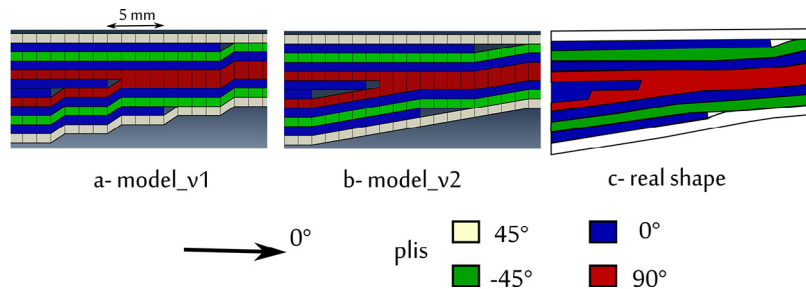


Fig. 12. Different meshing choices of the ply drop-off.

is under-predicted by the model; failure occurs at 58 kN for model_v1. It can also be noticed that in the simulation, the force reaches a plateau before failure which is slightly observed in the experimental curves. The modification of the mesh in model_v2 has improved the prediction of failure strength to 62.5 kN. Apart from this difference, results of both models are very similar during impact and CAI; that is why we will stick on model_v1 for the analysis of the results.

4.5. CAI displacement field

During the test, an Image Correlation system is used to monitor the impacted side of the specimen: displacement, strain and curvature fields of the surface are calculated from the data. In Fig. 13a, experimental out-of-plane displacement field is compared with the simulation (model_v1). Note that IC data are limited to a rectangle of $60 \times 130 \text{ mm}^2$ dimensions because the edges of the specimen are hidden by its supports. The predicted out-of-plane displacement field follows the same trend as the experiment. From the beginning, a local buckling appears in the thin section. This buckling is oriented toward the impacted side and is driven by the asymmetry of the taper. In the model, both sides of the specimen are moving toward the same direction like in the experiment. The buckling continues to grow throughout the test which explains the non-linearity observed in the force-displacement curve.

Surface strain field (Fig. 13b) is characterized by an important spatial variation. It results from both the thickness variation between the thin and the thick sections and the local buckling of the specimen. In the simulation, a higher longitudinal strain is observed at the beginning of the transition region compared to the IC measurement. This area is highly disturbed because the strain discontinuity induced by the ply drop-off is too important in the model.

4.6. CAI failure mechanism

The identification of the failure damage scenarios is quite complicated in a CAI test. Many failure modes can be possible: buckling, delamination propagation or fiber failure. The lack of information regarding internal plies does not help in the analysis and the use of simulation can therefore be of great help.

In this study, the behavior observed on the surface during the last moments of the test is similar to the simulation result. In both cases, a local buckling, highlighted in Fig. 14, occurs at one side of the specimen. The figure compares the increment of out-of-plane displacement before failure, when the force is between 65 kN and 69 kN in the experiment and between 55 kN and 57 kN in the simulation. There is an opening of sub-laminate which is associated, in the model, to a propagation of delamination in the transition region as shown in Fig. 15. This new delamination destabilizes the laminate and leads to the rupture of the specimen by failure of the 0° plies.

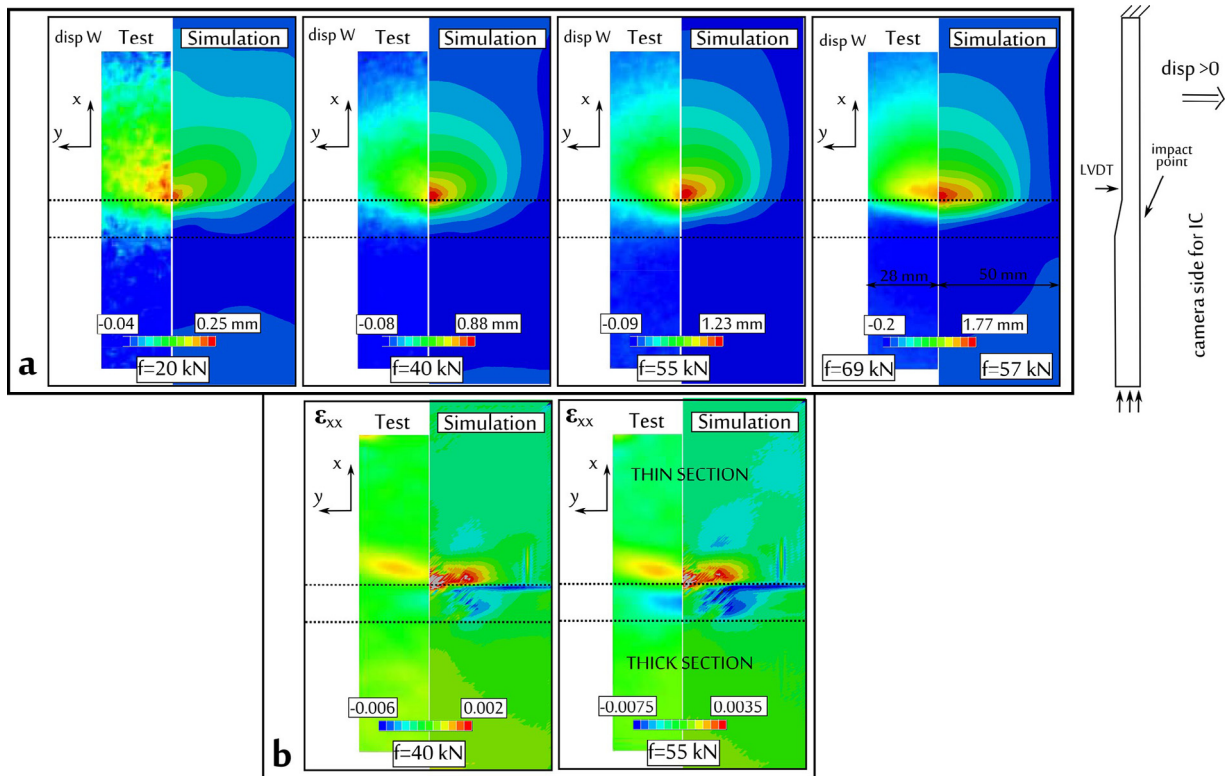


Fig. 13. Test and simulation comparison of CAI results (a) out-of-plane displacement and (b) specimen longitudinal strain at the impacted-face (model_v1).

Data regarding the strain field of 0° internal plies are required to corroborate the failure due to delamination. This information cannot be easily obtained from the test. To partially address the problem, the deformation can be computed from the surface strain and curvature field using Mindlin plate theory (Eq. (10)) [38]. Note that this equation is verified only in an undamaged area which is not completely satisfied in our case.

$$\epsilon_{xx}(z) = \epsilon_{xx}(z_{ext}) - (z - z_{ext}) * \frac{\partial^2 W}{\partial x^2}(z_{ext}) \quad \text{Eq. 11}$$

The computed strain of the continuous 0° ply near to the impacted face of the thin section is shown in Fig. 16. At 65 kN, the longitudinal compression failure strain of -0.0125 is reached at the impact point. There may be an initial fiber failure, however the values

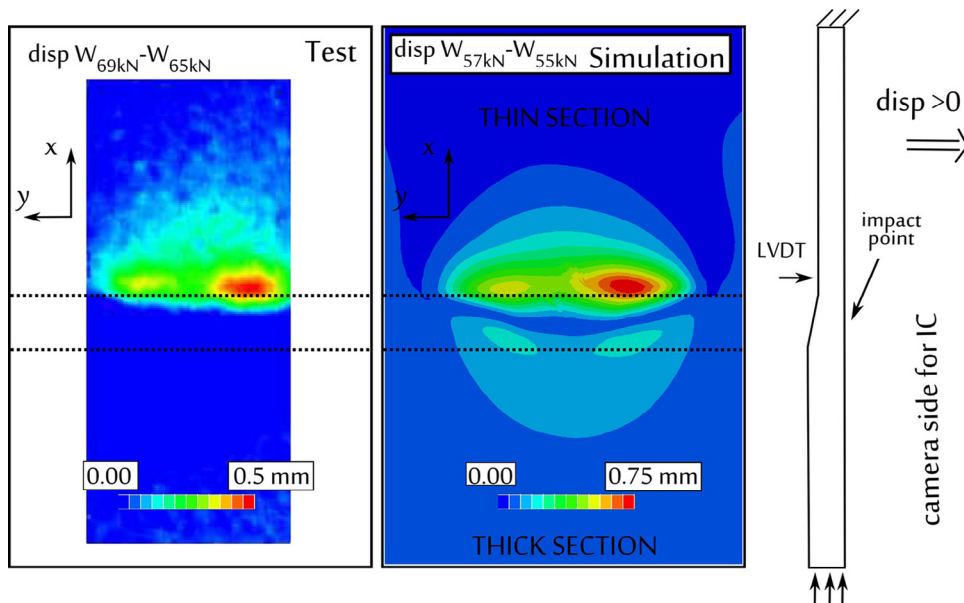


Fig. 14. Comparison of increment of out-of-plane displacement prior to specimen failure.

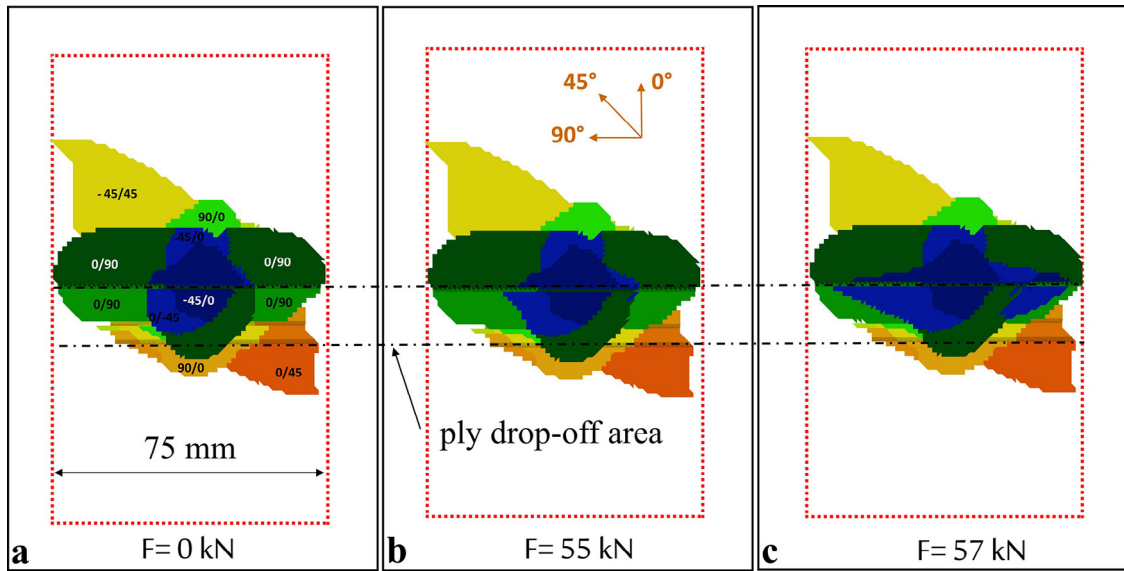


Fig. 15. Evolution of delamination at CAI failure.

should be taken with some reserve since the first interfaces are delaminated and the assumption of Mindlin plate theory is not satisfied locally. Fig. 16 shows the predicted strain field on both sides of the 0° ply before specimen failure, i.e. at 55 kN. Due to the delamination around the impact point, the group of plies is subjected to bending resulting in a more complex strain field. The strain is close to the longitudinal failure strain at some region of the ply.

Therefore, the prediction of external displacement field correlates with the test. A delamination does occur in the experiment, though it can be triggered by fiber failure. In the simulation, fiber failure is close to be reached when delamination triggers the specimen failure. Thus, the model is able to predict the complex failure scenarios despite that the failure load is under-predicted. The dialog between experiment and simulation has improved the understanding of the CAI test. The prediction of buckling and initiation of delamination in the transition region are important and can be improved by considering a more faithful modeling of the transition zone and a model with elements capable of simulating the bending of individual ply.

5. Conclusions

This paper reports a comprehensive numerical study about impact and CAI of tapered laminate. The use of DPM (Discrete Ply Model) approach has enabled to simulate both impact damage and residual compressive strength of the tested specimen. In the study, the resin pocket area is assumed to be damaged prior to impact. Consequently, any specific modification of the meshing approach is not required. Material laws take into account three types of damage: matrix cracking, delamination and fiber rupture. Impact simulation results are very close to the experimental data. Both the global contact force and delamination shape are well reproduced by the model. In CAI, the model under-estimates the failure load but predicts the CAI failure scenarios of the specimen. The model helps on the understanding of impact and CAI damage scenarios thanks to a successful experiment and simulation dialog.

Overall, this study has shown the robustness of the modeling approach for damage simulation of unidirectional laminate. Good results are obtained despite the presence of discontinuities related

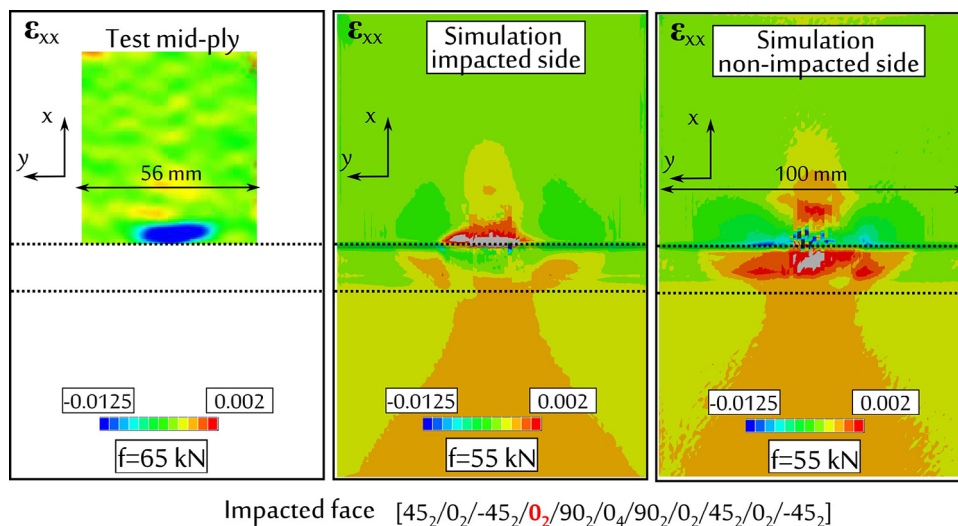


Fig. 16. Comparison of longitudinal strain of the first 0° internal ply.

to the ply drop-off. To the authors' knowledge, it is the first attempt to simulate both impact and CAI in presence of ply drop-off and from that perspective, the results are quite encouraging.

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