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Robust static output feedback sliding mode control design via an artificial stabilizing delay

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Abstract: It is well known that for uncertain linear systems, a static output feedback sliding mode controller can only be determined if a particular triple associated with the reduced order dynamics in the sliding mode is stabilizable. This paper shows that the static output feedback sliding mode control design problem can be solved for a broader class of systems if a known delay term is deliberately introduced into the switching function. Effectively the reduced order sliding mode dynamics are stabilized by the introduction of this artificial delay.

Keywords: Sliding mode control, output feedback, exponential stability, discretized sliding mode dynamics are stabilized by the introduction of this artificial delay.

1. INTRODUCTION

In many practical situations, all the states are not available to the controller. In some circumstances it is impossible or prohibitively expensive to measure all of the process variables. With this in mind, many authors have developed methods to control systems only using output feedback, of which one approach is the output feedback sliding mode control paradigm Edwards et al. [2003].

The idea developed in this paper is to broaden the class of systems for which a static output feedback based sliding mode controller can be developed based on a recent result from time delay systems. In Gu [2001], Fridman [2006], the authors show that for some systems, the presence of delay can have a stabilizing effect. This affords the possibility of taking a system which is not stabilizable by static output feedback without delay and finding a constant delay τ strictly greater than 0 such that the system is stable. In this case, a stabilizing delay is introduced into the dynamics to effect output feedback stability.

This design concept is not new. Several authors have considered this possibility Niculescu and Abdallah [2000], Niculescu et al. [2003], Michiels et al. [2004] and it has been shown that introducing a delay in an output feedback controller can stabilize a system which cannot be stabilized without delay. The novelty in this paper is in overcoming the output feedback stabilizability assumption of Edwards and Spurgeon [1995] in the design of sliding mode controllers by static output feedback. The authors propose a new switching function for robust control which contains an additional term which is linear in the delayed output developed in Seuret et al. [2007]. This is shown to be constructive in stabilizing the reduced order sliding mode dynamics. It is then shown that a sliding motion can be reached in finite time. Note that in this article the control law involves only a delay term and does not involve an amalgamation of delayed and non delayed terms.

The article is organized as follows. The second section presents the problem formulation. Section three formulates the definition of a new sliding function which contains an artificial delay. Section four deals with the exponential stabilization of non-delayed systems by a sliding mode controller including delay. In the last section, a numerical example demonstrates the design of the gains and the effect of the choice of the delay in the sliding mode controller.

Throughout the article, the notation $P > 0$ for $P \in \mathbb{R}^{n \times n}$ means that $P$ is a symmetric and positive definite matrix. For the sake of simplicity, a time-varying matrix $D(t)$ will be written as $D^t$. The symbol $I_n$ represents the $n \times n$ identity matrix. The notations $|.|$ and $\|.|\|$ refer to the Euclidean vector norm and its induced matrix norm, respectively. For any function $\phi$ from $C^1([-\tau; 0], \mathbb{R}^n)$, we denote $|\phi|_\tau = \sup_{s \in [-\tau, 0]} (|\phi(s)|)$. When there is no confusion, we write $x(t)$ as $x$.

2. PRELIMINARIES AND PROBLEM FORMULATION

Consider the linear uncertain system:

\[
\dot{x}(t) = A^t x(t) + B^t (u(t) + \psi(t, y)) \\
y(t) = C x(t) \tag{1}
\]

where $x \in \mathbb{R}^n$, $u \in \mathbb{R}^m$ and $y \in \mathbb{R}^p$ with $m < p < n$, corresponds to the state, control and output variables. The
function $\psi \in \mathbb{R}^m$ represents the matched disturbances. It is assumed there exists a known function $\Psi_2(t,y)$ such that:

$$||\psi(t,y)|| \leq \Psi_2(t,y)$$

(2)

The matrices $A^i \in \mathbb{R}^{n \times n}, B^i \in \mathbb{R}^{n \times m}, C \in \mathbb{R}^{p \times n}$ have appropriate dimension. It is also assumed that the input and the output matrices $B^i$ and $C$ are full rank and it is assumed $\text{rank}(CB^i) = m$, for all $t \in \mathbb{R}$. In addition and based on Edwards and Spurgeon [1995, 1998a], it is also assumed that there exists a change of coordinates $T$ such that, for all $t > 0$,

$$TB^i = \begin{bmatrix} 0 \\ B^i_t \end{bmatrix}$$

Then the system has the following representation:

$$\dot{x}(t) = \begin{bmatrix} A^i_{11} & A^i_{12} \\ A^i_{21} & A^i_{22} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ B^i_0 \end{bmatrix} (u(t) + \psi(t,y))$$

(3)

$$y(t) = \begin{bmatrix} 0 & T \end{bmatrix} x(t)$$

where $A^i_t \in \mathbb{R}^{(n-m) \times (n-m)}, B_0 \in \mathbb{R}^{n \times m}$ is non singular and $T \in \mathbb{R}^{p \times p}$ is an orthogonal matrix. The matrices have appropriate dimensions. It is assumed that, for all $i \in \{1, ..., N\}$, the pair of matrices $(A^i_0 + A^i_1, B_0)$ is controllable. The scalar functions $\lambda_i$ are such that:

$$\forall \ i = 1, ..., N, \lambda_i(t) \in [0, 1], \sum_{i=1}^{N} \lambda_i(t) = 1.$$

(4)

As it is possible to remove some uncertainties, the system is rewritten as:

$$\dot{x}(t) = \begin{bmatrix} A^i_{11} & A^i_{12} \\ A^i_{21} & A^i_{22} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ B^i_0 \end{bmatrix} (u(t) + \psi_0(t,y,u))$$

(5)

where the matched uncertainties are represented by:

$$\psi_0(t,y,u) = B^i_0 \left( \sum_{i=1}^{N} \lambda_i(t)(A^i_{22}x_2(t) + B^i_0u(t)) \right)$$

In Edwards and Spurgeon [1995] a sliding surface $S = \{x \in \mathbb{R}^n : FCx(t) = 0\}$ is proposed, where $F = F_2[K \ I_m]T^T$, $K \in \mathbb{R}^{n \times (p-m)}$ and $F_2 \in \mathbb{R}^{n \times m}$ is a non singular matrix. The sliding motion is governed by the choice of $K$. If a further coordinate change is introduced on the nonsingular transformation $z = \tilde{T}x$ with $\tilde{T}$ defined by:

$$\tilde{T} = \begin{bmatrix} I_{n-m} & 0 \\ 0 & C_1 \end{bmatrix}$$

with $C_1 = [0_{(p-m) \times (n-p)} I_{(p-m)}]$. Then as argued in Edwards and Spurgeon [1995], the dynamics of the reduced order sliding motion is governed by:

$$\dot{\tilde{x}}_1(t) = (A^i_{11} - A^i_{12}K_1) \tilde{x}_1(t) + A^i_{12} \tilde{x}_2(t) - A^i_{12}K_1 \tilde{x}_1(t) - \tilde{x}_2(t) = \begin{bmatrix} A^i_{21} & A^i_{22} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ B^i_0 \end{bmatrix} (u(t) + \psi(t,y,u))$$

(6)

To show the motivation of such a study, consider initially constant parameters, i.e. $\lambda_i(t) = 0$ for all $i = 1, ..., N$ and $t \geq t_0$. The fictitious system $(A^i_{11}, A^i_{12}, C_1)$ is assumed to be output stabilizable i.e., there exists a matrix $K$ such that the matrix $A^i_{11} - A^i_{12}K_1$ is Hurwitz. It is shown in Edwards and Spurgeon [1995] that a necessary condition for $(A^i_{11}, A^i_{12}, C_1)$ to be stabilizable is that the invariant zeros of $(A,B,C)$ lie in the open left half-plane. However the design of a gain $K$ such that the matrix $A^i_{11} - A^i_{12}K_1$ is Hurwitz is not always straightforward or even possible. Consider for instance the system (6) with:

$$A^i_{11} = \begin{bmatrix} 0 & -2 \\ 1 & 0 \end{bmatrix}, \quad A^i_{12} = \begin{bmatrix} -1 \\ 0 \end{bmatrix}, \quad C_1 = \begin{bmatrix} 0 & 1 \end{bmatrix}$$

The output feedback stabilization problem in this case becomes one of finding a scalar $k$ such that the matrix $\begin{bmatrix} 0 & -2 + k \\ 1 & 0 \end{bmatrix}$ has strictly negative eigenvalues, which is clearly not possible. In this situation some authors Bag et al. [1997], Edwards and Spurgeon [1998b], Edwards et al. [2003] have employed a compensator in order to stabilize the system. However these methods increase the order of the controller and have an associated computational overhead both in terms of design and implementation. The proposed method seeks to introduce an artificial delay in the system such that the system can be stabilized by static output feedback without the need to introduce a compensator.

3. DESIGN OF A NEW SLIDING MODE SURFACE WITH DELAY

In this section, the design of a new type of sliding surface will be discussed. The objective is to define a sliding surface of the form of $S$ but which introduces a delay in the reduced order dynamics. Consider

$$S' = \{x : FCx(t) + F_\tau Cx(t - \tau) = 0\}$$

(7)

where as before the matrix $F = F_2[K \ I_m]T^T$, $K \in \mathbb{R}^{n \times (p-m)}$ and $F_\tau = F_2[K_\tau \ 0_m]T^T$. Here without loss of generality, the matrix $F_2$ and $T$ are chosen as identity. In this definition $\tau$ is an artificial fixed and known delay which has to be chosen to stabilize the reduced order system and represents a design parameter. The existence of such a delay and constructive methods to choose it will be discussed in a latter section. Instead of $\tilde{T}$, consider the coordinate change $x \mapsto T_x x$:

$$\tilde{x}_1(t) = x_1(t), \quad \tilde{x}_2(t) = x_2(t) + KC_1 x_1(t) + K_\tau C_1 x_1(t - \tau)$$

By construction the sliding function associated with $S'$ is $s(t) = \tilde{x}_2(t)$. This leads to

$$\dot{\tilde{x}}_1(t) = (A^i_{11} - A^i_{12}K_1) \tilde{x}_1(t) + A^i_{12} \tilde{x}_2(t) - A^i_{12}K_1 \tilde{x}_1(t) - \tilde{x}_2(t) = \begin{bmatrix} A^i_{21} & A^i_{22} \end{bmatrix} x(t) + \begin{bmatrix} 0 \\ B^i_0 \end{bmatrix} (u(t) + \psi(t,y,u))$$

(8)

Remark 1. Note that the system (8) is a particular delay system. Since the delay $\tau$ is artificially introduced, it is known and can be chosen to improve the stability of the closed-loop system.

Remark 2. Note that the delayed sliding motion produces a more complicated system since the resulting equation contains several delayed terms and two different delays, $\tau$ and $2\tau$. 
Note that the last four lines of the previous equation only depend on the output information and thus the following output feedback control law can be defined:
\[
\begin{align*}
    u(t) &= -(B_{pg})^{-1} \{(A_{11}^{0} + K_{1} C_{1}^{0}) x_{2}(t) \\
    &+ K_{C} C_{1}^{0} \tilde{x}_2(t) + \nu + (A_{12}^{0} + K_{C} C_{1}^{0}) \tilde{x}_1(t) \\
    &+ e^{\varepsilon \tau_{i}}(K_{C} C_{1}^{0}) \tilde{x}_1(t) - R_{p} Q_{2} - e^{\varepsilon \tau_{i}}(K_{C} C_{1}^{0}) \tilde{x}_1(t) \\
    &- K_{C} C_{1}^{0} \tilde{x}_1(t) - 2\tau - G_{1} \tilde{x}_2(t)\}
\end{align*}
\]

where \( G_{1} \) is a Hurwitz matrix. The closed loop system satisfies the following equations:
\[
\begin{align*}
    \dot{x}_1(t) &= (A_{11}^{0} - A_{11}^{0} K_{C} C_{1}^{0}) \tilde{x}_1(t) \\
    &- A_{12}^{0} K_{C} C_{1}^{0} \tilde{x}_1(t) - 2\tau + (A_{11}^{0} + K_{C} C_{1}^{0}) \tilde{x}_1(t) \\
    \dot{\tilde{x}}_2(t) &= G_{1} \tilde{x}_2(t) + (A_{12}^{0} + K_{C} C_{1}^{0}) \tilde{x}_1(t) \\
    &+ K_{C} C_{1}^{0} \tilde{x}_1(t) - \nu + \psi_{1}(t, y, u)
\end{align*}
\]

where
\[
\psi_{1}(t, y, u) = \sum_{i=1}^{N} \lambda_{i}(t) [K_{C} C_{1}^{0} \tilde{x}_2(t) + K_{C} C_{1}^{0} \tilde{x}_1(t) - K_{C} C_{1}^{0} \tilde{x}_2(t) - K_{C} C_{1}^{0} \tilde{x}_1(t)] + \sum_{i=1}^{N} \lambda_{i}(t) \\
= [K_{C} C_{1}^{0} \tilde{x}_2(t) - K_{C} C_{1}^{0} \tilde{x}_1(t) - K_{C} C_{1}^{0} \tilde{x}_1(t) - K_{C} C_{1}^{0} \tilde{x}_2(t)] + B_{0} \psi_{0}(t, y, u)
\]

As \( \psi_{1} \) only depends on \( t, y, u \), there exists a positive function \( \Psi_{0}(q) \) such that:
\[
\|\psi_{1}(t, y, u)\| \leq B_{0}\|\Psi_{2}(t, y, u) + \Psi_{21}(t, y, u)\|
\]

Now define the discontinuous sliding function \( \nu \) by:
\[
\nu(t, y, u) = \begin{cases} 
    \frac{Q_{2} \tilde{x}_2(t)}{\|Q_{2} \tilde{x}_2(t)\|} & \text{if } \tilde{x}_2(t) \neq 0 \\
    0 & \text{otherwise}
\end{cases}
\]

where \( Q_{2} \) is a symmetric positive definite matrix in \( R_{n \times m} \) and
\[
\rho(t, y, u) = \|B_{0}\|\|\Psi_{2}(t, y, u) + \Psi_{21}(t, y, u)\| + \delta
\]

where \( \delta \) is a positive scalar gain.

**Remark 3.** Note that the control law (9) does not have a heavy computational overhead.

## 4. STABILIZATION OF THE CLOSED LOOP SYSTEM

This section focusses on the stability of the whole system (10). In particular, it needs to be established that \( \tilde{x}_2 = 0 \) in finite time, i.e. a sliding motion is achieved.

### 4.1 Exponential Stability Condition

**Theorem 1.** System (10) is exponentially stable for given output feedback gains \( K \) and \( K_{C} \) with decay rate \( \alpha \) if there exist \( P_{1} > 0, P_{2}, P_{3}, S_{p} = S_{p}^{T}, Q_{p}, R_{pq} = R_{pq}^{T}, p, q = 0, \ldots, N \) in \( R_{n \times m} \times R_{n \times m} \) and \( Q_{2} > 0 \) in \( R_{n \times m} \) which satisfy LMI’s (12) and (13) with \( h = \tau / N \) and for all \( i = 1, \ldots, N \):

\[
\begin{align*}
    \begin{bmatrix} 
        \Xi_{i} & D_{s} & D_{s} \\
        -R_{d} - S_{d} & 0 & 0 \\
        * & -3S_{d} & 0
    \end{bmatrix} &< 0 \\
    \begin{bmatrix} 
        P_{1} & \bar{Q} & \bar{R} + S
    \end{bmatrix} &> 0
\end{align*}
\]

where the matrix \( \Xi_{i} \) is given by:
\[
\begin{bmatrix} 
    \Psi_{i} P_{T} \begin{bmatrix} \nu_{i} \alpha_{\lambda_{i}} - Q_{N} \\
        0
    \end{bmatrix} & -S_{N} \\
    * & -S_{N}
\end{bmatrix}
\]

where \( \forall i = 1, \ldots, N \) and \((k, l) \in [1, 2]^{2} \):

\[
\Psi_{i} = \begin{bmatrix} 
    Q_{0} + Q_{0}^{T} & 0 \\
    0 & P_{T} \begin{bmatrix} \alpha_{\lambda_{i}} - I_{N} \\
        \alpha_{\lambda_{i}} - I_{N}
    \end{bmatrix}
\end{bmatrix}
\]

Proof. As in Seuret et al. [2004], consider new variables \( \tilde{x}_{1o}(t) = \tilde{x}_1(t)e^{\nu t} \) and \( \tilde{x}_{2o}(t) = \tilde{x}_2(t)e^{\nu t} \). The new closed-loop system satisfies the following equations:
\[
\begin{align*}
    \dot{\tilde{x}}_{1o}(t) &= \begin{bmatrix} 
        A_{11}^{1} - A_{11}^{1} K_{C}^{1} + \alpha I_{n \times m} \\
        -e^{\varepsilon \tau_{i}}(K_{C} C_{1}^{0}) \tilde{x}_1(t) - R_{p} Q_{2} - e^{\varepsilon \tau_{i}}(K_{C} C_{1}^{0}) \tilde{x}_1(t) \\
        \end{bmatrix} \\
    \dot{\tilde{x}}_{2o}(t) &= (G_{1} + \alpha I_{n \times m}) \tilde{x}_{2o}(t) + (A_{12}^{1}) \\
    &+ K_{C} C_{1}^{0} \tilde{x}_{1o}(t) - e^{\varepsilon \tau_{i}}(\psi_{1}(t, y, u)) \\
    &+ e^{\varepsilon \tau_{i}}(K_{C} C_{1}^{0}) \tilde{x}_{1o}(t) - \nu(t, y, u)
\end{align*}
\]

Consider the Lyapunov-Krasovskii functional:
\[
V_{o}(t) = V_{1o}(t) + V_{2o}(t),
\]

where \( V_{2o}(t) = \tilde{x}_{2o}(t) Q_{2} \tilde{x}_{2o}(t) \) and:
\[
\begin{align*}
    V_{1o}(t) &= \tilde{x}_{1o}(t) P_{1} \tilde{x}_{1o}(t) + \frac{1}{2} \tilde{x}_{1o}(t) \int_{t}^{\xi} Q_{1}(\xi) \tilde{x}_{1o}(t + \xi) d\xi \\
    &+ \int_{t}^{\xi} \tilde{x}_{1o}(t + \xi) S(\xi) \tilde{x}_{1o}(t + \xi) d\xi \\
    &+ \int_{t}^{\xi} \tilde{x}_{1o}(t + \xi) R(\xi) R(\xi) \tilde{x}_{1o}(t + \xi) d\xi
\end{align*}
\]

where \( P_{1} > 0, Q(\xi) \in R_{n \times m} \times R_{n \times m}, R(\xi, s) = R_{T}(\xi, s), S(\xi) \in R_{n \times m} \times R_{n \times m}, \) and \( Q, R, S \) are continuous matrix functions. From Gu et al. [2003] (p. 185) \( V_{1o} \) is positive definite if the LMI (13) holds. Then the proof follows along the lines of Fridman [2006] using a descriptor representation, Fridman and Shaked [2002], and discretization Gu [2001]. Differentiating the Lyapunov functional \( V_{1o} \) along the trajectories of (15) and using the convexity of functions \( \lambda_{i} \) leads to:
An ideal sliding motion takes place on the plane \( S \) expressed in an appropriate representation to apply the inequality can be established as in Fridman [2006]:

\[
\dot{\xi}(t) = \int_{0}^{t} Q(\xi)\dot{\xi} x_{1\alpha}(t + \xi) d\xi
\]

Integrating the expression above by parts, the following equality can be established as in Fridman [2006]:

\[
\check{V}_{1\alpha}(t) = \sum_{i=1}^{N} \lambda_i(t) \left\{ \xi^T(t) \Xi_{1\alpha} \xi(t) + \int_{-\tau}^{0} Q(\xi) \check{\xi}_{1\alpha}(t + \xi) d\xi \right\}
\]

where \( \xi(t) = \text{col}(\check{\xi}_{1\alpha}(t), \check{\xi}_{1\alpha}(t - \tau)) \) and \( \Xi_{1\alpha} \) has the form in (14) and the matrix function \( R(s, \xi) \) is chosen to be linear along the trajectory of (15b). The last term is thus negative. Applying Proposition 5.21 from Gu et al. [2003] to (22) and using the convexity of \( \lambda_i \), it can be concluded that \( V(t) < 0 \) if LMI’s (12) hold for all \( i = 1, \ldots, N \).

4.2 Reachability to the sliding manifold in finite time

**Corollary 1.** An ideal sliding motion takes place on the surface \( \check{z}_2(t) = 0 \) in the domain

\[
\Omega = \{(\check{z}_1, \check{z}_2) \in [-\tau, t) \rightarrow R^{n-m} \times R^{m} : \max_{i,j=1,\ldots,N} (\|A_{1i} + KC_{1j}\| + \|K_{ij}C_{1j}\|)|\check{z}_1| \leq \delta < \eta \}
\]

where \( \eta \) is a small scalar satisfying \( 0 < \eta < \delta \).

**Proof.** Consider the following Lyapunov function \( V_{s}(t) = \xi_{1\alpha}^T(t)Q_{s}\xi_{1\alpha}(t) \). By differentiating \( V_{s} \) along the trajectories of (10b), it follows that:

\[
\dot{V}_{s}(t) = \xi_{1\alpha}^T(t)Q_{s}\xi_{1\alpha}(t) + \xi_{1\alpha}^T(t)\left[ A_{1\alpha} - KC_{11}\right]\xi_{1\alpha}(t) - \nu + \psi(t, y, u) = \|B_{20}\|\Psi_{2}(t, y, u) + \Psi_{21}(t, y, u) + \delta, \text{ the following inequality holds:}
\]

\[
\dot{V}_{s}(t) \leq -\eta \sqrt{V_{s}(t)}
\]

This concludes the proof.

**Remark 4.** As usual, the problem of designing the output feedback gain is not straightforward. Moreover LMI (12) is not an appropriate form for synthesis purposes because the gains \( K \) and \( K_{ij} \) appear in different ways in \( \Xi_{a} \) than in \( (K_{11}C_{11})^T Q_{s} \) and \( (K_{12}C_{11})^T Q_{s} \). Congruence and other classical LMI transformations will probably not facilitate
constructive conditions. A constructive method at this time is to test the stability of the closed-loop system for a given set of values of $K$ and $K_\tau$ as shown in the following example.

5. EXAMPLE

Consider the non-delayed system (5) with:

\[
\begin{align*}
A_{t1} &= \begin{bmatrix} 0 & -2 \\ 1 & 0.1 \end{bmatrix}, & A_{t2} &= \begin{bmatrix} -1 \\ 0.1(1 - \cos(x_1)) \end{bmatrix}, \\
A_{21} &= \begin{bmatrix} -0.1 & -1 \\ 1 & 1 \end{bmatrix}, & A_{22} &= \begin{bmatrix} 1 \\ 1 \end{bmatrix}, \\
B_\tau^T &= \begin{bmatrix} 0 & 0 & 1 \end{bmatrix}, & C &= \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}.
\end{align*}
\]

As in Edwards and Spurgeon [1995], this system is not output stabilizable using traditional static (i.e., non-delayed output feedback). The objective here is to design the controller (9) with appropriate gains $K$, $K_\tau \in \mathbb{R}$ and an artificial delay $\tau$ such that the closed-loop system is exponentially stable with decay rate $\alpha$.

5.1 Design of the output feedback

This section proposes a method to obtain the optimal controller $(K, K_\tau, \tau)$. The idea is to test if, for a set of values of $K$ and $K_\tau$, the LMIs from Theorem 1 have a solution and if it is possible to find the delay which ensures the greatest exponential decay rate.

Theorem 1 can only be satisfied when $K$ lies in the interval $[-4; 2]$ and $K_\tau$ in $[0; 4]$. For each value of the gains $K$ and $K_\tau$, an optimization process is used to obtain the best value of $\alpha$ by tuning $\tau$ upwards from zero until the LMIs are not satisfied.

The decay rate is $\alpha = 0.445$. Theorem 1 also ensures for $N = 6$ that the same gains exponentially stabilize the system (5) with a decay rate $\alpha = 0.538$ with $\tau = 0.425$.

Remark 5. Starting from $N = 2$, the computation of the conditions from Theorem 1 become very heavy. The optimization problem has not been tested for $N \geq 3$.

5.2 Simulation results

![Fig. 2. Simulation results for $K = -1.7, K_\tau = 2.8$ and $\tau = 0.425$s](image)

In the results which follow system (5) is controlled using (9) with $K = -1.7, K_\tau = 2.8$ and $\tau = 0.425$.

Figure 2 shows the state, the input and the sliding function. The state converges exponentially to $x(t) = 0$ with an exponential decay rate $\alpha = 0.538$. The sliding function converges to $s(t) = 0$ in finite time. The evolution of the control signal is also shown in Figure 2.

The robustness with respect to the delay is shown in Figure 3. Assuming that the gains $K$ and $K_\tau$ are still $K = -1.7$ and $K_\tau = 2.8$, Figure 3 presents several simulations for different delays. It can be seen that the delay has an influence on the stability. For small delays ($\tau = 0.01$ or $\tau = 0.9$), system (5) is unstable. However, when the delay is sufficiently close to the optimal delay ($\tau = 0.3$ or $\tau = 0.6$), system (5) becomes stable. It can also be seen in figures 3 that the decay rates for $\tau = 0.3$ or $\tau = 0.6$ are less than in the optimal delay case shown in figure 2. This proves the efficiency of the proposed method.

6. CONCLUSION

A new sliding mode controller has been suggested for systems for which finding a traditional static output feedback sliding mode controller is not possible. The controller introduces a stabilizing delay in the closed-loop system. The controller is simple and does not require heavy computation and it ensures robust exponential stability of the closed-loop system. An example has been used to
Fig. 3. Simulation results for $K = -1.7$, $K_\tau = 2.8$ and different values of $\tau$
demonstrate a method to design the gains and the delay of the controller. It also shows that this controller still stabilizes the system even when system has vertices which have invariant zeros on the right side of the complex plane.

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