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# Turbulent vortex dipoles in a shallow water layer 

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This paper describes an experimental study on turbulent dipolar vortices in a shallow water layer. Dipoles are generated by an impulsive horizontal jet, by which a localized three-dimensional turbulent flow region is created. Dipole emergence is only controlled by the confinement number $C=\sqrt{Q} / H^{2} t_{\text {inj }}$ whereas the jet Reynolds number $\operatorname{Re}=\sqrt{Q} / v$ has no influence in the studied range $50000<\operatorname{Re}<75000$ ( $H$ is the water depth, $v$ is the kinematical viscosity, $Q$ the injected momentum flux and $t_{\mathrm{inj}}$ the injection duration). When $C>2$, the flow becomes quasi-two-dimensional and a single vortex dipole emerges in most cases. By qualitative observations and application of particle image velocimetry, the main dipole features have been determined. The shallow water dipoles are characterized by the simultaneous presence of several scales of turbulence: A quasi-two-dimensional main flow at large scale and three-dimensional turbulent motions at small scale. A vertical circulation takes place in the dipole front. A theoretical model is presented and compared to experimental results. The three-dimensional turbulence production occurs mainly in the frontal circulation. A good agreement has been found between the model prediction and the measurements for the velocity evolution. © 2004 American Institute of Physics. [DOI: 10.1063/1.1762912]

## I. INTRODUCTION

At the beginning of the twentieth century, Lamb ${ }^{1}$ and Chaplygin ${ }^{2}$ have achieved well-known theoretical studies on two-dimensional invisible flows based on the Euler equation. In particular, they have detailed the structure of coherent two-dimensional vortices. One of them is the dipole, which consists of two closely packed counter-rotative circulations. It has the remarkable property of transporting momentum and mass like a solid body does. Observations on geophysical flows have revealed the presence, at very varied dimensions, of large horizontal vortices. At large scale, the satellite imagery has clearly shown the existence of coherent vortices in the atmosphere and the oceans. Most of these vortices, which possess long lifetimes, are monopolar vortices. ${ }^{3}$ Nevertheless, dipolar vortices could have an important mixing role due to their ability to transport matter over long distances. At smaller scale, in the near-shore zone for instance, dipoles have been observed in the head of rip-currents ${ }^{4}$ or in tidal channels. ${ }^{5}$ But, whereas theoretical models assume the strict two-dimensionality of the flow and the velocity fields, geophysical flows generally possess vertical components associated to three-dimensional motions. When these motions are negligible, the flows are considered quasi-twodimensional (Q2D).

Laboratory experiments on quasi-two-dimensional flows have confirmed the appearance of dipolar vortices. The quasi-two-dimensionality is obtained by submitting the flow to specific forcing. They are mainly the density stratification,
the background rotation, the magnetic forces and the thickness of the fluid domain itself.

Flòr and van Heijst ${ }^{6}$ and Voropayev et al. ${ }^{7}$ have both experimentally studied the evolution of an impulsive jet in a stratified fluid. After the gravitational collapse due to the stratification, the flow becomes approximatively twodimensional and a vortex dipole eventually emerges. The dipole so created is not systematically symmetric. Depending on the type of injection, laminar or turbulent, the relationship between the vorticity and the stream function can be respectively linear or nonlinear (sinh-relationship). The nonlinearity of the relationship is in discrepancy with the LambChaplygin model. In their statistical-mechanics approach of characterizing steady-state structures in two-dimensional (2D) flows, based on the searching for the most probable state, Montgomery and Joyce ${ }^{8}$ found evidence for the validity of this sinh-relationship for such 2D flow structures. Despite of the nonlinear relationship observed in the dipoles in stratified fluid, very good agreement has been found with the main features of the Lamb-Chaplygin dipole, by comparing the size, position of maximum vorticity or the translation speed. Moreover Voropayev et al. shown the self-similarity of the dipole during time.

Experiments in rotating fluids have revealed the emergence of two-dimensional vortices. ${ }^{9}$ In particular, Velasco Fuentes and van Heijst ${ }^{10}$ have experimentally studied the propagation of dipolar vortices on a topographic $\beta$-plane. Measurements have shown the nonlinearity of the vorticity/ stream function relationship.

Dipoles have also been observed by Nguyen Duc and Sommeria. ${ }^{11}$ In their experiments, the flow twodimensionality is obtained by submitting a horizontal layer of mercury to a vertical uniform magnetic field. The dipoles were found to be in good agreement with the theoretical model: Circular shape and vorticity-stream function relation with a constant slope (or without maximum). Possible interactions have been observed between dipoles, which leads to exchange of vortices.

Vortex dipoles can also emerged in the wake of a cylinder towed through a soap-film. ${ }^{12}$ In this case, the extreme thinness of the fluid domain leads to the flow twodimensionality.

Thus, experiments in stratified ${ }^{6,7}$ or rotating fluids ${ }^{9,10}$ and soap films ${ }^{12}$ have revealed the quasi-two-dimensional behavior of the turbulence, characterized by the emergence of large coherent vortices. The dipoles frequently observed in these flows have been widely studied and compared with theoretical 2D model. As shown recently by experiments carried out on mixing layers ${ }^{13,14}$ and grid turbulence, ${ }^{15}$ forced turbulent shallow flows can also develop a Q2D dynamics, characterized by emergence of large coherent vortices. These studies have revealed the simultaneous presence of large scale Q2D vortices and small scale 3D turbulence induced by bottom friction. Shallow layers are then defined as fluid layers where the coherent structure size is much larger than the layer depth. However, any experimental characterization of the transition from deep to shallow layers is lacking: The transition point where two-dimensional processes start dominating the turbulence behavior has not been clearly identified. Moreover, any study of freely evolving coherent turbulent vortices in shallow water is still lacking. Due to their implication in turbulent geophysical processes (mixing processes and transports of mass, momentum or sediments), it is of great importance to know the vortices evolution and their interaction with the solid bottom.

Our study consists into two parts. First, we aim at identifying the transition from a deep to a shallow water layer in a test case: The vortex dynamics produced by an impulsive jet in a homogeneous water layer. Second, we focus our attention on the free evolution of vortex dipoles in a shallow water layer.

Qualitative observations and quantitative measurements have been carried out with the experimental setup described in Sec. II. The dimensional analysis of impulsive turbulent jets is proposed in Sec. III. The influence of the dimensionless numbers at large scale is exposed in Sec. IV. The observations on the dipoles are summarized in Sec. V. From these observations, a theoretical model is proposed in Sec. VI and compared with experimental results. Finally, a discussion of the results and conclusions are proposed in Sec. VII.

## II. EXPERIMENTAL SETUP

A sketch of the experimental setup is presented in Fig. 1. Experiments have been performed in a rectangular channel ( $8 \mathrm{~m} \times 3 \mathrm{~m}$ ). In this channel, we inject a small amount of fluid through a horizontal cylindrical nozzle (diameter 8 mm ) immersed in water at rest. During our experiments, three


FIG. 1. Sketch of the experimental setup.
parameters are modified: The depth, the injection rate and the injection duration. The depth H varies from 0.21 to 0.35 m , the injection rate from $2.5 \cdot 10^{-4}$ to $7.5 \cdot 10^{-4} \mathrm{~m}^{-3} / \mathrm{s}$ and the injected duration from 1 to 20 s . The injected fluid is taken outside of the channel and is strictly the same as the receiving fluid. The injection rate is held constant during the injection duration.

Qualitative informations have been obtained by seeding the flow with pliolite particles (size $300 \mu \mathrm{~m}$, density 1023.6 $\mathrm{kg} \cdot \mathrm{m}^{-3}$ ) and using a laser sheet illumination described here after.

Velocity measurements have been performed by particle image velocimetry (PIV). PIV is a nonintrusive velocity measurement technique which provides instantaneous velocity fields. It consists in analyzing the displacements of particles added to the flow.

The illumination is provided by an argon laser with power 8 W (model "coherent innova 70-4 A"). The light sheet is produced with an oscillating mirror. The total angular spread in water can reach $50^{\circ}$, but it can be adjusted to optimize luminosity in the field of view. This horizontal laser sheet can be quickly moved to cover three horizontal slices, the lower one at 5 cm above the bottom, the middle one at half-depth and the upper one 3 cm below the free surface. A CCD camera (SMD 1 M60) with a $1024 \times 1024$ resolution is used. It captures frames with an adjustable frequency up to 60 Hz . A numerical shutter is available on each frame and gives access to higher velocities. The observed areas are relatively large $\left(2.5 \times 2.5 \mathrm{~m}^{2}\right)$. Frames sequences are recorded from the injection until the flow goes out the observation area. Typical duration of recording is about 300 s .

The step following the frame grabbing is the correlation process, performed by CIV (correlation imaging velocimetry). CIV is an advanced imaging velocimetry implementation of direct-cross correlation PIV that relies on generalized pattern matching by direct cross-correlation of pattern boxes between image pairs. The software CIV provides the two components of the velocity $u$ and $v$ on each point of the meshgrid, and the velocity gradients in each direction $\partial u / \partial x$, $\partial v / \partial x, \partial u / \partial y$, and $\partial v / \partial y$ calculated with a centered scheme. From these gradients, the vorticity and the horizontal divergence can be directly calculated: $\omega=\partial u / \partial y$ $-\partial v / \partial x$ and $\partial w / \partial z=-\partial u / \partial x-\partial v / \partial y$. Next the stream
function $\psi$ is calculated by numerically solving the following Poisson-type equation: $\omega=-\nabla^{2} \psi$. The global uncertainties on velocity measurements are inferior to $2 \%$ and those obtained on differential quantities are two times greater, about $4 \%$. For a complete review on CIV features, the reader is referred to Refs. 16 and 17.

The seeding of the flow is of great importance for the quality of PIV measurements. The particles need to accurately follow the local flow. It means that the particles should have a density very close to that of the fluid and a small size to avoid perturbations. The seeding density (particle concentration) is governed by the need to have roughly 0.05 particles per pixel. This density should be homogeneous in the whole observed region. In our measurements, we have used pliolite particles, with a size about $300 \mu \mathrm{~m}$ and a density of $1023.6 \mathrm{~kg} \cdot \mathrm{~m}^{-3}$. Salt has been added to the water to match the particle density.

## III. DIMENSIONAL ANALYSIS

In order to minimize surface excitations by the injection, we use a small injector nozzle $d \ll H$ ( $d$ the nozzle diameter and $H$ the depth). We study the jet at distance $x$ large in comparison with the nozzle diameter $d$. In this zone $x$ $>25 d$, the turbulent jets are self-similar: ${ }^{18}$ The jet diameter $d$ is not a relevant parameter and the expansion rate can be considered constant. For a turbulent impulsive jet, the two relevant parameters are the injected momentum flux $Q^{7}$ and the injection duration $t_{\mathrm{inj}}$. Thus, four independent parameters describe our experiments:
(i) $\quad v$, the kinematic viscosity $\left[\mathrm{m}^{2} \cdot \mathrm{~s}^{-1}\right]$;
(ii) $Q$, the injected momentum flux given by $Q=(U S)^{2} / S=U^{2} S=\frac{1}{4} U^{2} \pi d^{2}\left[\mathrm{~m}^{4} \cdot \mathrm{~s}^{-2}\right] ;$
(iii) $H$, the water depth [m];
(iv) $t_{\mathrm{inj}}$, the injection duration $[\mathrm{s}]$;
where $U$ is the injection speed and $S$ the nozzle section.
From these parameters, two dimensionless numbers can be defined. First, the jet Reynolds number Re, based on the injected momentum flux, is given by

$$
\begin{equation*}
\operatorname{Re}=\frac{\sqrt{Q}}{v} \tag{1}
\end{equation*}
$$

We have called the second dimensionless number $C$ the confinement number, defined by

$$
\begin{equation*}
C=\frac{\sqrt{Q}}{H^{2}} t_{\mathrm{inj}} \tag{2}
\end{equation*}
$$

The confinement number $C$ characterizes directly the vertical confinement applied on the impulsive jet: The greater $C$ is (which corresponds to large injected volume or small depth), the stronger the confinement applied is.

The time (observation time from the injection) $t$ is also a preponderant parameter. The dimensionless time $t^{*}$ is defined from the injection duration: $t^{*}=t / t_{\text {inj }}$.


FIG. 2. Evolution of the correlation product for $C=0.9$ (symbols $\bigcirc$ ), $C=1.7$ (symbols + ) and $C=8$ (symbols $\cdot$ ).

## IV. INFLUENCE OF THE VERTICAL CONFINEMENT

The first step of the present experimental study is to quantify the influence of $C$ on the flow, we determine the flow structuration as a function of $C$. Structuration is evaluated by computing the correlation product between the velocity fields on three horizontal slices: the upper one 3 cm below the free surface, the middle one at half depth and the lower one 5 cm above the bottom. The correlation product is calculated as follows:

$$
\operatorname{corr}=\sum_{i, j} \frac{u_{1} u_{2}+v_{1} v_{2}}{\sqrt{u_{1}^{2}+v_{1}^{2}} \sqrt{u_{2}^{2}+v_{2}^{2}}} \frac{1}{n b_{\text {points }}},
$$

where $\left(u_{1}, v_{1}\right)$ and $\left(u_{2}, v_{2}\right)$ are the velocity components given at each point of the PIV mesh grid on two different slices, $i$ and $j$ describe the mesh grid constituted by $n b_{\text {points }}$.

The correlation product corr is equal to 1 when the velocity fields $\left(u_{1}, v_{1}\right)$ and ( $u_{2}, v_{2}$ ) are totally identical. The arithmetical average is computed between the values obtained for the upper and middle slices and for the middle and the lower slices. Examples of the correlation product evolution for $C=0.9,1.7$, and 8 are shown in Fig. 2. The three experiments have different acquisition durations due to the different dynamics which can be more or less propagative. For the cases $C=0.9,1.7$, and 8 the durations are, respectively, 200, 407, and 127 s . In order to compare the evolution of the correlation product on the same graph, the time has been normalized by the acquisition duration. For $C=0.9$, the correlation product increases very slowly from 0.61 to 0.67 , the structuration remains weak. For $C=1.7$, the correlation product increases from 0.58 to 0.8 and reach a nearly constant value around 0.8 . It means that the turbulent patch generated with $C=1.7$ develops progressively in a structured flow on the whole depth. For the most confined case $C=8$, the initial value of the correlation product is greater than 0.8 . It increases to reach 0.9 until the end of the acquisition. For $C=8$ the structuration has occurred before that the flow goes in the PIV acquisition area.


FIG. 3. Maximum of the correlation product versus $C$ for $\operatorname{Re}=50000$ and $\mathrm{Re}=75000$.

In order to quantify the evolution of the flow structuration with respect to the confinement number, we have compared the maximum value reached by the correlation product for each experiment. This maximum is called Mcorr. The evolution of Mcorr versus the confinement number $C$ is plotted in Fig. 3. We note a direct influence of $C$ on Mcorr while Re has not a significant role. When $C$ is small $(C<1)$, the confinement is too weak to generate a structuration of the flow and the turbulence remains fully three-dimensional (3D). From $C=2, M c o r r$ remains superior to 0.8 , the flow is structured over the depth. The corresponding velocity fields indicate that these structured flows are vortex dipoles. Note that the correlation product never reaches 1 . It means that the flow is not perfectly structured, vertical variations remaining even in the most confined experiments. Most of the impulsive jets generated with $C<2$ do not develop a structured flow but we have observed episodic emergence of dipoles for $C=1.7$.

Thus, three regimes can be identified in the jet development. When $C<1$, the confinement does not influence the jet, the water layer is deep. A transition occurs when $1<C$ $<2$, where the confinement starts to act on the jet evolution, but the flow is not systematically structured. When $C \geqslant 2$, the "shallow water behavior" appears: this is characterized by the flow structuration in large horizontal dipole. We do not have identified any influence of the jet Reynolds number.

## V. OBSERVATIONS ON DIPOLES

Vortex dipoles are systematically formed when $C \geqslant 2$, and can also appear in the transition zone $1<C<2$. The experimental results presented here describe a dipole obtained for $C=3.2$ and $\operatorname{Re}=50000$, which features are representative of all other dipoles we observed in our shallow water experiments.


FIG. 4. Particle streaks of a dipole obtained with $C=3.2$ and $\operatorname{Re}=50000$ at $t^{*}=22.1$ (upper) and $t^{*}=42$ (lower).

## A. Qualitative observations

Qualitative observations have been carried out by adding fluoresceine or particles in the flow. The pictures shown in Figs. 4 and 5 have been taken for a dipole obtained with $C$ $=3.2$ and $\mathrm{Re}=50000$.

The particles streaks (Fig. 4) reveal the main dipole structure with two counter-rotative circulations. The upper


FIG. 5. Picture of a dipole obtained with $C=3.2$ and $\mathrm{Re}=50000$ at $t^{*}=35.8$.
picture taken at $t^{*}=22.1$ shows the presence of a perturbed zone in the front of the dipole. This zone is characterized by small-scale turbulent motions. In the latter picture $\left(t^{*}\right.$ $=42$ ), the dipole size has increased. The perturbed zone remains in the dipole front, but we also note the presence of 3D turbulent motion in the dipole main flow. A preliminary hypothesis can be done here: 3D small scale turbulence is produced in the frontal region and then transported in the main Q2D dipole flow.

A picture with fluoresceine added to the injected fluid (Fig. 5) confirms the presence of a perturbed zone in the dipole front. Small scale vortices take place all over the frontal boundary.

## B. Velocity and vorticity fields

The first quantitative measurements are done on the velocity fields. These velocity fields are obtained by the PIV system presented in Sec. II.

Figure 6 shows velocity fields and corresponding vorticity distributions, respectively, on upper, middle, and lower slices of the dipole obtained with $C=3.2$ and $\mathrm{Re}=50000$. Measurements were done at $t^{*}=40.3,40.4$, and 40.5 . The shifts correspond to the time necessary to move the light sheet.

The dipolar structure with two counter-rotative circulations is easily identified on the velocity fields. The corresponding vorticity fields are two patches of opposite sign. However, the dipole structure is not totally identical on the three slices, there is a vertical variation of the flow. The main difference between the velocity fields occurs in the front of the dipole. On the upper slice, the fluid in this frontal zone flows out of the dipole, while on the lower slice the flow clearly comes inside the dipole. In the middle slice, the frontal flow is not structured, many small-scale motions are observed. These observations reveals the presence of a vertical circulation in the front of the dipole, which is detailed in the following section. Moreover, the flow is not laminar, small shear zones associated to small vortex structures remain inside the dipole. This feature is confirmed on the vorticity distributions which show numerous patches of various signs. This small-scale turbulence, mainly concentrated in the frontal zone, has been previously observed during the qualitative visualizations.

## C. Vertical motions

The dipole flow presents a vertical variation in the front of the dipole. The comparison between the velocity fields in the upper and the lower slices suggests the presence of a vertical circulation in the front of the dipole. These observations are confirmed by studying the horizontal divergence of the velocity fields.

The divergence of the velocity field is nil because of the flow incompressibility. Thus, from the horizontal velocity fields, we can compute the gradient of vertical velocity over the vertical direction as follows:
$\partial w / \partial z=-\partial u / \partial x-\partial v / \partial y$.
Figure 7 shows the distribution of $\partial w / \partial z$ in the dipole on
the upper and lower slices, respectively, at $t^{*}=58.2$ and at $t^{*}=58.4$. We note first that the peak value of $\partial w / \partial z$ is half the peak value of the vertical vorticity. Then, the horizontal divergence can not be neglected. Even if the main flow takes place in the horizontal plane, the velocity field is not 2D. We note that the distribution of $\partial w / \partial z$ is not uniform: the strongest vertical velocity gradients are located in the front of the dipole. The comparison between the upper and the lower slices reveals the presence of a vertical circulation. The PIV slices intercept the upper and lower part of this circulation which takes place all along the front of the dipole and all through the depth. The role of this frontal circulation in the production of small-scale turbulence observed in the whole dipole is analyzed in Sec. VI A.

## D. $\omega-\psi$ relationship

The Lamb-Chaplygin model of purely 2D dipole assumes a linear relationship between the vorticity $\omega$ and the stream function $\psi$ in the dipole. The experiments carried out on dipoles ${ }^{6,10,11}$ have shown several different relationships between $\omega$ and $\psi$. In most cases, the $\omega / \psi$ relationship is nonlinear and the derivative of the function $\omega=f(\psi)$ has singular maxima at the vortex cores, but no local maximum elsewhere.

The stream function is only defined for purely 2D flows. The dipoles observed in shallow water are characterized by significant vertical motion. However, we can assume a scaleseparation: A main 2D flow at large length and time scales and 3D turbulent motions at small length and time scales. The main flow is a 2D dipole, which omega-psi relationship can be analyzed. In order to minimize the perturbation due to 3D motions, the vorticity and the stream function are computed from the average of the three velocity fields (on the upper, middle, and lower slices) measured for the case $C=3.2$ and $\mathrm{Re}=50000$. The so obtained omega-psi relation is presented in the left part of the Fig. 8. The plot dispersion reveals the presence of small-scale turbulence in the dipole main flow. The small-scale vortices of 3D turbulence provide numerous patches of vorticity which directly induce strong noise in the vorticity signal. The stream function which is defined from two successive spatial integrations of the vorticity is less affected by the presence of small scale motion. In order to evaluate the omega-psi relationship of the 2D main flow and to avoid perturbations due to small-scale turbulence, we have averaged the vorticity values obtained for each stream function calculation. The result is plotted on the right part of the Fig. 8. It allows to identify the nonlinear relationship between the vorticity and the stream function of the dipole main flow. The shape of the omega-psi relation is different from the sinh-relation observed in stratified or magnetically driven flows. We can note that the two branches $\omega>0$ and $\omega<0$ which each represent one half of the dipole are more spaced than in a sinh-relation. It means that the two vortices are less linked and are separated by a nearly irrotational flow. It reveals an increase of fluid entrainment, probably caused by the vertical circulation at the dipole front. This observation can also be relied to the numerical simulations of dipole in stratified fluid carried out by Beckers


FIG. 6. Dipole velocity field (left) and vorticity field (right) for $C=3.2$ and $\operatorname{Re}=50000$ : upper figure at $t^{*}=40.3$ on the upper slice, middle figure at $t^{*}$ $=40.3$ on the middle slice, lower figure at $t^{*}=40.3$ on the lower slice (the black arrows represent the jet axis).
et al. ${ }^{19}$ For the higher Reynolds number simulations, dipoles (which remain laminar) are found to be less compact, with a kink in the vorticity profile. For increasing Reynolds number, entrainment of nearly irrotational fluid into the dipole becomes more effective in forming less compact dipoles, because diffusion is less effective in smoothing the vorticity profile. Such a mechanism can explain the appearance of a
kink in the vorticity profile in our turbulent dipole experiments.

## E. Summary of the observations

The present experiments have highlighted the main features of dipoles in a shallow water layer. These dipoles are


FIG. 7. Horizontal divergence field (left for upper slice and right for lower slice, units $\mathrm{s}^{-1}$ ), at $t^{*}=58.2$ and $t^{*}=58.4, C=3.2$ and $\operatorname{Re}=50000$ (the black arrow represents the jet axis).
quite different from their counterparts observed in Q2D flows. Firstly, the dipoles are turbulent: 3D turbulence is advected by the Q2D main flow. These small scale perturbations are responsible for the dispersion of the $\omega / \psi$ relationship. Secondly, vertical motion remains significant during the dipole evolution. These motion are mainly concentrated in a vertical circulation which takes place in the dipole front all over the depth.

## VI. THEORETICAL MODEL

## A. Hypothesis

We have used the previous observations to model the dipole evolution and to understand the presence of smallscale turbulence in the main flow. Our model aims at describing the evolution of a dipole in linear translation, of speed $U$ and radius $a$ with a vertical circulation in the frontal zone (Fig. 9).

The equilibrium of the boundary layer corresponds to the balance between the diffusion and advection times of the momentum. This leads to

$$
\frac{\delta^{2}}{v} \approx \frac{H}{U} .
$$

Then, the boundary layer thickness $\delta$ can be estimated


FIG. 8. Vorticity/stream-function relationship for $C=3.2, \operatorname{Re}=50000$ and $t^{*}=58.3$, direct computation (left figure) and averaged vorticity computation (right figure).

$$
\delta \approx \sqrt{v \frac{H}{U}}
$$

By considering the characteristic values $U=0.02 \mathrm{~m} / \mathrm{s}$, $v=10^{-6} \mathrm{~m}^{2} \cdot \mathrm{~s}^{-1}$ and $H=0.3 \mathrm{~m}$, the boundary layer Reynolds number $\operatorname{Re}_{\delta}$ is about 80, which is far from the critical value $\operatorname{Re}_{\delta}=300$. Then, the boundary layer is considered stable. We assume that the frontal circulation of the dipole is the only responsible of small-scale turbulence production.

The momentum $P$ (divided by $\rho$ ) and the kinetic energy $E$ (divided by $\rho$ ) are, respectively, evaluated by

$$
\begin{align*}
& P \approx H a^{2} U  \tag{3}\\
& E \approx H a^{2} U^{2} \tag{4}
\end{align*}
$$

## B. Energy dissipation

The kinetic energy dissipation can be written

$$
\begin{equation*}
\frac{d E}{d t}=\frac{d U P}{d t}=P \frac{d U}{d t}+U \frac{d P}{d t} \tag{5}
\end{equation*}
$$



FIG. 9. Schematic representation of the dipole structure.


FIG. 10. Evolution of reduced momentum and reduced kinetic energy. For $C=8(\operatorname{Re}=50000), C=4.8(\operatorname{Re}=50000), C=3.2(\operatorname{Re}=50000), C=2.2$ $(\operatorname{Re}=50000), C=1.7(\operatorname{Re}=75000)$, and $C=1.7(\operatorname{Re}=50000)$.

## 1. Estimation of $P(d U / d t)$

This term is associated to the turbulence production which occurs in the frontal circulation, characterized by its thickness $H$ and its "active" volume $a H^{2}$. The energy dissipation rate in the circulation can be written as the product of the volumic dissipation $U^{3} / H$ by the active volume

$$
\begin{equation*}
P \frac{d U}{d t} \approx-\frac{U^{3}}{H} a H^{2} \approx-U^{3} a H \approx-\frac{E U}{a} . \tag{6}
\end{equation*}
$$

The radius $a$ can be expressed from Eq. (3): $a$ $\approx P^{1 / 2} U^{-1 / 2} H^{-1 / 2}$. Then Eq. (6) becomes

$$
\begin{equation*}
P \frac{d U}{d t} \approx-U^{5 / 2} P^{1 / 2} H^{1 / 2} \tag{7}
\end{equation*}
$$

## 2. Estimation of $U(d P / d t)$

This term is nil if the momentum is constant. A remarkable property of the 2 D dipoles is to transport momentum as a solid body does. Significant vertical motions have been observed in the shallow water dipoles. It is of great interest to know how these long-lifetime vortices are able to transport momentum.

TABLE I. $\left(\operatorname{Re}_{f} / / \operatorname{Re}_{a}\right)^{1 / 4}$ versus $C$ and Re.

| $C$ | $\mathrm{Re}_{\text {inj }}$ | $\left(\mathrm{Re}_{f}^{3} / \mathrm{Re}_{a}\right)^{1 / 4}$ initial | $\left(\mathrm{Re}_{f}^{3} / \mathrm{Re}_{a}\right)^{1 / 4}$ final | Final time (s) |
| :--- | :---: | :---: | :---: | :---: |
| 8 | 50000 | 25.4 | 8.6 | 127 |
| 4.8 | 50000 | 11 | 8.9 | 113 |
| 3.2 | 50000 | 17.2 | 5.9 | 263 |
| 2.2 | 75000 | 25.5 | 10.9 | 340 |
| 1.7 | 50000 | 15.6 | 10.7 | 407 |
| 1.7 | 50000 | 25.5 | 9.6 | 483 |

If $P$ is not constant, its dissipation is mainly caused by the bottom friction. It can be estimated by

$$
\begin{equation*}
\frac{d P}{d t} \approx v \int \frac{\partial v}{\partial z} d r^{2} \tag{8}
\end{equation*}
$$

The shear $\partial v / \partial z$ can be approximated by $U / \delta$, where $\delta$ is the boundary layer thickness. Equation (8) becomes

$$
\begin{equation*}
\frac{d P}{d t} \approx v \frac{U a^{2}}{\delta} \tag{9}
\end{equation*}
$$

Then Eq. (9) can be written

$$
\begin{equation*}
\frac{d P}{d t} \approx-v^{1 / 2} U^{3 / 2} H^{-1 / 2} a^{2} \tag{10}
\end{equation*}
$$

which leads to

$$
\begin{equation*}
U \frac{d P}{d t} \approx-v^{1 / 2} U^{5 / 2} H^{-1 / 2} a^{2} \tag{11}
\end{equation*}
$$

## 3. Comparison of both contributions

From Eqs. (6) and (11), the total kinetic energy dissipation becomes

$$
\begin{equation*}
\frac{d E}{d t} \approx-U^{3} a H-v^{1 / 2} U^{5 / 2} H^{-1 / 2} a^{2} \tag{12}
\end{equation*}
$$

The contributions of both terms can be evaluated by computing their ratio

$$
\begin{equation*}
\frac{P \frac{d U}{d t}}{U \frac{d P}{d t}}=U^{3} a H v^{-1 / 2} U^{-5 / 2} H^{1 / 2} a^{-2}=U^{1 / 2} H^{3 / 2} a^{-1} v^{-1 / 2} \tag{13}
\end{equation*}
$$

If we introduce the parametrization of shallow water flows proposed by Dolzhanskii et al., ${ }^{20}$ two different Reynolds numbers can be defined. On one hand, the internal Reynolds number characterizes the horizontal structure

$$
\begin{equation*}
\operatorname{Re}_{a}=\frac{U a}{v} \tag{14}
\end{equation*}
$$

On the other hand, the external Reynolds number takes into account the bottom friction

$$
\begin{equation*}
\operatorname{Re}_{f}=\frac{U}{\lambda L}=\frac{U H^{2}}{2 a v} \tag{15}
\end{equation*}
$$

where $L$ is the characteristic horizontal lengthscale and $\lambda$ is the bottom friction coefficient given by $\lambda=2 v / H^{2}$ for a thin fluid layer.

Equation (13) can be written

$$
\begin{align*}
\frac{P \frac{d U}{d t}}{U \frac{d P}{d t}} & =\left(\frac{U H^{2}}{2 a v}\right)^{3 / 4}\left(\frac{U a}{v}\right)^{-1 / 4}  \tag{16}\\
& =\left(\frac{\operatorname{Re}_{f}^{3}}{\operatorname{Re}_{a}}\right)^{1 / 4} \tag{17}
\end{align*}
$$

Two different regimes are possible:
(i) If $\left(\operatorname{Re}_{f}^{3} / \operatorname{Re}_{a}\right)^{1 / 4}>1\left(\operatorname{Re}_{f}^{3} \gg \operatorname{Re}_{a}\right)$, the term $P d U / d t$ dominates, the frontal circulation has a preponderant influence, the momentum dissipation can be neglected.
(ii) If $\left(\operatorname{Re}_{f}^{3} / \operatorname{Re}_{a}\right)^{1 / 4}<1\left(\operatorname{Re}_{f}^{3}<\operatorname{Re}_{a}\right)$, the momentum cannot be considered constant. This regime has been characterized in the parametrization of bottom friction in the shallow water equations, see for instance, Ref. 21.

## C. Velocity and radius evolution

In the case where $\left(\operatorname{Re}_{f}^{3} / \operatorname{Re}_{a}\right)^{1 / 4}>1$, the momentum is considered constant. Then evolution laws for $U$ and $a$ can be deduced from expression (7)

$$
\begin{equation*}
U \approx\left(\frac{P}{H}\right)^{1 / 3}\left(t-t_{0}\right)^{-2 / 3}, \quad a \approx\left(\frac{P}{H}\right)^{1 / 3}\left(t-t_{0}\right)^{1 / 3} \tag{18}
\end{equation*}
$$

This time dependence of the dipole radius is similar to the one obtained in the experiments carried out on dipoles in stratified fluid by Voropayev et al. ${ }^{7}$

## VII. COMPARISON WITH EXPERIMENTAL RESULTS

## A. Momentum measurements

The first step of the model validation is to quantify the momentum evolution during the dipole translation and to compare it with the kinetic energy evolution.

Six nearly symmetric dipoles in linear translation have been studied. From the three horizontal slices, the velocity fields $(u, v)$ are obtained all along the dipole evolution. The momentum $P_{i}$ and the kinetic energy $E_{i}$ (divided by $\rho$ ) are computed on the slice $(i)$ as follows:

$$
\begin{aligned}
& P_{i}=\sqrt{\left(\sum_{S} u d S\right)^{2}+\left(\sum_{S} v d S\right)^{2}} \\
& E_{i}=1 / 2 \sum_{S}\left(u^{2}+v^{2}\right) d S
\end{aligned}
$$

where $S$ is the PIV meshgrid surface and $d S$ the surface element. The average values $P$ and $E$ are computed between the three slices. By dividing $P$ and $E$ by their respective maxima, the reduced momentum $P^{*}$ and the reduced kinetic energy $E^{*}$ are obtained.

Figure 10 shows the compared evolutions of $P^{*}$ and $E^{*}$ for six different dipoles, respectively, represented by sym-


FIG. 11. Evolution of reduced velocity on upper slice (symbols $*$ ), middle slice (symbols + ), and lower slice (symbols $\bigcirc$ ), and comparison with theoretical model $U^{*}=A\left(t^{*}-t_{0}^{*}\right)^{-2 / 3}$ (solid line).
bols $\bigcirc$ and + . The dimensionless time is defined by $t^{*}=t / t_{\mathrm{inj}}$. The initial time $t^{*}=0$ is the beginning of the PIV acquisition.

We first note that, in all cases, the reduced momentum is better conserved than reduced kinetic energy. However, the momentum dissipation is never negligible. This dissipation is weak in the most confined case ( $C=8$ in Fig. 10) but remains significant. Thus, we have shown that, during the propagation of dipoles in shallow water, the momentum is not strictly conserved. The momentum dissipation is associated to the bottom friction, which contribution in the kinetic energy dissipation is evaluated by the term $U(d P / d t)$ in Eq. (5).

As shown previously, the contributions of both terms $U(d P / d t)$ and $P(d U / d t)$ in the kinetic energy dissipation can be evaluated by the ratio $\left(\operatorname{Re}_{f}^{3} / \mathrm{Re}_{a}\right)^{1 / 4}$. We compute the values of $\mathrm{Re}_{f}, \mathrm{Re}_{a}$ and $\left(\mathrm{Re}_{f}^{3} / \mathrm{Re}_{a}\right)^{1 / 4}$ for each dipole, at the initial and final time of the acquisition. The characteristic speed $U$ is determined on the dipole axis and the radius $a$ is defined as the distance between the vorticity extrema. These calculations are exposed in the Table I.

Two observations can be done from these results:
(i) $\left(\operatorname{Re}_{f}^{3} / \operatorname{Re}_{a}\right)^{1 / 4}$ is systematically greater than 1 . It means that the term $P(d U / d t)$ dominates $U(d P / d t)$ in Eq. (17). In first approximation, we can neglect the term


FIG. 12. Evolution of the constant $A$ versus the confinement number $C$ compared with hyperbolic evolution $A=36.8 C^{-1}$.
$U(d P / d t)$ in the kinetic energy dissipation, which corresponds to considering the momentum constant during the dipole evolution.
(ii) $\left(\operatorname{Re}_{f}^{3} / \mathrm{Re}_{a}\right)^{1 / 4}$ decreases during time. The regime considered here, where the momentum dissipation is neglected and the turbulence production in the frontal circulation is the main responsible of kinetic energy dissipation, will progressively degenerate in a different regime where both contributions are of the same order.
The assumption $P=$ cste allows to write evolution laws for the characteristic velocity and radius in Eq. (18).

## B. Model validation

In order to validate our model, its predictions are compared with the experimental results for the dipole velocity and radius. The evolution laws are given by

$$
\begin{equation*}
U \approx A\left(t-t_{0}\right)^{-2 / 3}, \quad a \approx B\left(t-t_{0}\right)^{1 / 3}, \tag{19}
\end{equation*}
$$

where $A$ and $B$ are constant and $t_{0}$ is, respectively, defined by $U\left(t_{0}\right)=U_{0}$ where $U_{0}$ is the initial dipole velocity and $a\left(t_{0}\right)=a_{0}$ where $a_{0}$ is the initial dipole radius. The reduced velocity $U^{*}$ and the reduced radius $a^{*}$ are obtained by dividing the measured values $U$ and $a$ by the initial values: $U^{*}=U / U_{0}$ and $a^{*}=a / a_{0}$.

## 1. Evolution of characteristic velocity

Figure 11 presents the evolutions of the characteristic velocity measured on the three slices compared to the model prediction. In every case, we can note a good agreement between the model prediction and the experimental results. The constant $A$ takes respectively the following values: 3.4, 10, 12, 16, 23, and 20. In Fig. 12, the experimental values $A_{\text {exp }}$ are plotted versus $C$ and compared to a hyperbolic evolution $A=36.8 C^{-1}$. The reduced characteristic velocity of turbulent shallow water dipoles can be expressed as follows:

$$
\begin{equation*}
U^{*}=36.8 C^{-1}\left(t^{*}-t_{0}^{*}\right)^{-2 / 3} . \tag{20}
\end{equation*}
$$

## 2. Evolution of radius

The dipole radius is computed from the vorticity field obtained from the PIV velocity field. By identifying at each instant the vorticity extrema positions, we deduce the distance between the circulation centers, called $a$.

Figure 13 presents the evolutions of the reduced radius measured on the three slices compared to the model prediction. The constant $B$ takes, respectively, the values: $0.6,0.2$, $0.1,0.2,0.2$, and 0.05 .

The model predictions for the dipole radius are less accurate than for the velocity evolution. The dipole radius, identified as the distance between the circulation centers, does not increase systematically. In the cases $C=4.8$ and $C$ $=1.7$, it tends to decrease during time. If we compare the vorticity distribution obtained at $t^{*}=25.6$ and $t^{*}=131.5$ in the last case ( $C=1.7$ ) in Fig. 14, we observe that the circulation centers get closer during time while the dipole lateral extension increases mainly by lateral diffusion. Unfortunately, the total dipole lateral extension cannot be correctly measured because of the difficulty to define a boundary to the dipole flow.

We can also observe an oscillation of the dipole radius around its evolution. These oscillations are not identical on the three slices, there is a phase shift. The vorticity axes do not remain parallel during time. Dipoles are submitted to horizontal oscillations, which lead to conical shapes alternatively oriented downward and upward. This phenomenon is less evident in the most confined case $C=8$.

## VIII. CONCLUSION

Laboratory experiments aimed at studying the vortex dynamics generated by impulsive jets have been performed in shallow water layer. Our aim was to identify a transition from fully 3D behavior (deep fluid layer) to Q2D behavior (shallow water layer). This transition has been studied by imposing a progressive vertical confinement on an axisymmetric horizontal turbulent impulsive jet. The jet evolution has been studied depending on two dimensionless parameters: the Reynolds number Re and the confinement number $C$. They are defined by

$$
\operatorname{Re}=\frac{\sqrt{Q}}{v}, \quad C=\frac{\sqrt{Q}}{H^{2}} t_{\mathrm{inj}}
$$

where $v$ is the kinematic viscosity, $Q$ the injected momentum flux, $H$ the water depth and $t_{\mathrm{inj}}$ the injection duration. The development of horizontal impulsive jets is only controlled by the confinement number $C$ whereas the Reynolds number has no influence in the studied range. When $C$ is inferior to 1 , the impulsive jet development is a typical three-dimensional turbulence decay, the water layer is deep. Between $C=1$ and $C=2$, the confinement starts to act on the flow, vertical motions are progressively damped. Beyond $C=2$, the impulsive jet develops into vortex dipole, which size is much larger than the fluid depth, the water layer can be considered shallow.

A detailed study of turbulent dipoles in shallow water layer have been carried out. The velocity and vorticity fields show the relative homogeneity of the dipoles over the depth


FIG. 13. Evolution of reduced radius on upper slice (symbols $*$ ), middle slice (symbols + ), and lower slice (symbols $\bigcirc$ ), and comparison with theoretical model $a=B\left(t^{*}-t_{0}^{*}\right)^{1 / 3}$ (solid line).
and the quasi-two-dimensionality of the main flow. However, the dipoles are turbulent: 3D turbulent motions are transported by the Q2D main flow. Observations have revealed the presence of a vertical circulation in the dipole front. The boundary layer on the solid bottom is stable and the frontal circulation is the only responsible of small-scale turbulence production. The contributions of bottom friction and frontal circulation in the kinetic energy dissipation are controlled by the ratio $\left(\operatorname{Re}_{f}^{3} / \operatorname{Re}_{a}\right)^{1 / 4}\left(\operatorname{Re}_{a}=U a / v\right.$ and $\operatorname{Re}_{f}=U H^{2} / 2 a v$ where $U$ is the characteristic velocity and $a$ the dipole radius). Measurements have shown that $\left(\operatorname{Re}_{f}^{3} / \operatorname{Re}_{a}\right)^{1 / 4}$ is greater than 1 for every dipole observed in shallow water. Thus, even if the momentum is not strictly conserved, the effects of momentum dissipation by bottom friction can be neglected. Two
evolution laws for the velocity $U$ and radius $a$ of the dipoles can be deduced from the model: $U=A\left(t-t_{0}\right)^{-2 / 3}$ where $t_{0}$ is defined by $U\left(t_{0}\right)=U_{0}$ with $U_{0}$ the initial speed and $A$ is a constant, and $a=B\left(t-t_{0}\right)^{1 / 3}$ where $B$ is a constant and $a\left(t_{0}\right)=a_{0}$ where $a_{0}$ is the dipole initial radius. The comparison with experimental results shows a good agreement for the velocity evolution. Such comparison is more difficult for the radius evolution because of the difficulty to carry out accurate measurements.

Thus, we have characterized the conditions necessary to the formation of dipoles in a shallow water layer and detailed the flow structure. However, these experimental results lead us to several questions, in particular the role of the solid bottom in the formation of a vertical circulation in the dipole


FIG. 14. Vorticity fields at $t^{*}=25.6$ and $t^{*}=131.5$, half-depth, $C=1.7, \operatorname{Re}=75000$ (the black arrow represents the jet axis).
front. Preliminary qualitative observations have been carried out at small scale on laminar dipoles with a slip condition on the bottom (addition of a heavier fluid layer). These observations have revealed the quasi-two-dimensional structure of the flow, without vertical circulation at the dipole front. ${ }^{22}$

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