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Dynamic and quasi-static lateral compression tests of ceramics tubes

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Abstract: In order to investigate the deformation of ceramics tube in lateral compression, a series of experiments was carried out at dynamic and static rates. It was observed that the load and deflection when fracture occurred in dynamic tests were greater than those obtained in static tests. However, a difference between Young's moduli obtained dynamically and statically was not found. By using a primary curved beam theory, Young's modulus and the fracture stress of ceramics tube could be derived from the results of experiments. The Young's modulus showed reasonable agreement with that obtained from tensile tests. The fracture stress also showed the value close to the bending strength of the ceramics tube.

1. INTRODUCTION

Recently, fine ceramics have been thought the third material following metals and fiber reinforced plastics (FRP) and attracted a great deal of researchers' attention. However, ceramics are generally brittle and very hard. Therefore, manufacturing specimens for material testing is difficult and costly compared with the case of metals. This is one of the obstacle to obtain the mechanical properties of ceramics. The lateral compression test of circular tubes sandwiched by two rigid plates appears to be one of the useful testing methods to investigate the strength of circular tubes. The specimen for this test is a tube just cut off from a long tube and the testing arrangement is quite simple. In addition, the influence of friction between a specimen and rigid compression plates, as observed in ordinary axial compression tests, is negligible because the specimen has line contacts with compression plates. Therefore, it is very convenient if we can use this test to obtain the mechanical properties of a ceramics tube such as Young's modulus and fracture stress.

With respect to the deformation of circular tube in lateral compression, a number of analytical and experimental studies have been carried out[1-8]. One of the early analytical studies is the study by Deruntz and Hodge[2] in 1963, in which they considered the deformation of a circular tube with four
plastic hinges. Reid & Reddy[4] proposed a more accurate model in which two curved beams were adopted instead of two plastic hinges and linear strain hardening was taken into account. A number of numerical analyses using finite element method (FEM) were also performed to examine the large plastic deformation[5], the mechanism of fracture occurred in relatively brittle circular tubes[6, 7] and crack initiation in filament-wound composite circular tubes[8].

Dynamic behaviour of circular tubes in lateral compression was investigated from the point of view of the application to a shock absorber, by using thin-walled tubes of mild steel and aluminium[9], of relatively brittle aluminium alloys[10] and FRP circular cylinder[11]. This behaviour was also discussed being compared with the deformation of a simple structure made of two pre-bent plates[12,13]. In others, the collapse of braced circular tubes[14] and the dynamic responses of one- and two-dimensional ring systems[15,16] under impact loads were also investigated.

In this paper, lateral compression tests for ceramic tubes were carried out at dynamic and quasi-static rates to examine the effect of loading rate and specimen length on Young's modulus and fracture stress of ceramics. Static tensile tests were also performed to determine the basic mechanical properties of ceramics. Young's modulus and fracture stress, which were estimated from the results of lateral compression tests, were compared with the values obtained from tensile and bending tests.

2. EXPERIMENTS

2.1 Material and Specimen

The material tested is Mullite (Al₂O₃: 53 wt.%, SiO₂: 41 wt.%) ceramic tube, sold on the market, with 6 mm in outer diameter, 1 mm in wall thickness and 300 mm in length. The tube was just cut to make specimens for dynamic and static lateral compression tests. Three different lengths of specimen, 4, 7 and 10 mm, were adopted to check the effect of specimen length. In order to obtain basic mechanical properties of tube material, static tensile tests were performed by using a simple method which was recently proposed by Nojima and Iiyama[17]. Young's modulus, \(E\), Poisson's ratio, \(\nu\), and tensile strength, \(\sigma_t\), obtained from tensile tests are shown in Table 1 with bending strength, \(\sigma_b\), and density, \(\rho\), which are from a catalogue.

2.2 Static lateral compression test

The setting for static lateral compression tests is shown in Fig.1. Since the dimension is small, both the load and deflection are very small such, 300 N and 0.05 mm, respectively. Therefore, a special load cell and a "deflection cylinder" were newly prepared. The deflection was obtained from the longitudinal strain of the deflection cylinder. The testing speed is about \(2 \times 10^3\) mm/min.

More than 10 specimens were used for each kind of test.

2.3 Dynamic lateral compression test

Dynamic tests were performed by using split Hopkinson pressure bar system, as shown in Fig.2. The input, output and striking bars are made of A6063 aluminium alloy tube with outer diameters of 16

| Table 1 Density and mechanical properties of Mullite tube. |
|---------------|-------|-------|-------|-------|
| \(E\) (GPa)  | \(\nu\) | \(\sigma_t\) (MPa) | \(\sigma_b\) (MPa) | \(\rho\) (kg/m³) |
| 114          | 0.25  | 90    | 157   | 2.6×10³ |

Fig.1 Setting for static lateral compression tests.

V₀ = 1.2 m/sec

Fig.2 Layout of split Hopkinson pressure bar used for dynamic lateral compression test.
and 13 mm, respectively. The striking bar was accelerated by a spring up to about 1.2 m/sec. The speed of striking bar just before it hit the input bar was measured by a simple device which consists of semiconductor laser and photo-diode. Stress waves propagating in the input and output bars were picked up by semiconductor strain gauges glued on these bars. The pulses of these waves were recorded in a digital oscilloscope after passing through bridge boxes and pre-amplifiers. Typical stress waves of a specimen with 10 mm in length are shown in Fig.3. The incident wave has nearly rectangular shape, although its rising time is about 30 μsec. The length of the striking bar is 250 mm, the velocity of the stress wave in the aluminium alloy rod, \( C_a \), is about 5000 m/sec and the duration of the incident wave is about 100 μsec as observed in Fig.3. Therefore, the one-dimensional theory for the propagation of stress waves may be used to estimate the average load and deflection of a tubular specimen. Here, the applied load, \( P \), and deflection, \( \delta \), of a ceramics tube were determined by the following equations:

\[
P = \frac{EA}{2} (\varepsilon_i + \varepsilon_R + \varepsilon_T) = EA\varepsilon_T, \quad \delta = C_o \int_0^L (\varepsilon_i - \varepsilon_R - \varepsilon_T) \, dt = 2C_o \int_0^L (\varepsilon_i - \varepsilon_T) \, dt \quad \ldots (1)
\]

where \( \varepsilon_i = -(\varepsilon_r - \varepsilon_T) \), \( E \) and \( A \) are the Young's modulus and cross-sectional area of the input and output bars, \( C_o \) is the velocity of the stress wave given by \( C_o = \sqrt{E/\rho} \), and \( \varepsilon_i, \varepsilon_r, \varepsilon_T \) are strains obtained from the incident, reflected and transmitted waves, respectively.

3. EXPERIMENTAL RESULTS AND DISCUSSIONS

3.1 Load-Deflection Curves

Two typical load-deflection curves for specimens with lengths of 4 and 10 mm obtained from static tests are shown in Fig.4. The load increases gradually at first and then increases linearly up to the maximum load. Four peaks, as seen in Fig.4, appeared in most load-deflection curves. At the first and second peaks, fracture occurred at the locations A, shown in Fig.7(a), i.e. the tubular specimen was separated vertically into two parts. The third and fourth peaks in Fig.4 correspond to the fractures which occurred at locations B (see Fig.7(a)). This means that the specimen is finally divided into four pieces. These four pieces of specimen are almost same size, as shown in Fig.5. The reason why the fracture is relatively stable, may be that the deflection cylinder acts as a kind of stabilizer, although it was inserted to measure the deflection of the specimen.
Fig. 6 shows the result of dynamic tests with a 10 mm specimen. The load increases almost linearly up to the maximum load, as in the static results. Since the slope of this region is almost the same as in static tests, Young's modulus may not be affected by loading rate. However, the maximum load in dynamic tests is much greater than that obtained from static tests, see Fig. 4. This may be caused by the effect of loading rate. After the first peak, two or three more peaks are observed. These peaks may correspond to the vertical and horizontal fractures in the static tests. The deformation speed is also shown in Fig. 6 and is equal to \( \frac{\partial \delta}{\partial t} = 2C_0(e_1 - e_0) \), see equ. (1). Although a drop is observed close to the deflection of \( \delta = 0.014 \text{ mm} \), the speed until the first peak of the load is roughly 0.6 m/sec. If the striking bar has the same cross-sectional area as the input bar, the particle velocity in the input bar becomes half that of the striking bar. Therefore, this is quite reasonable because the speed of the striking bar is about 1.2 m/sec, as mentioned before.

3.2 Estimation of Young's modulus and fracture stress

When we think about the deformation of a circular tube in lateral compression as shown in Fig. 7(a), the deformation of a quadrant is usually considered from equilibrium and symmetry of the deformation. Fig. 7(b) shows the system of forces and moments acting on one quadrant during deformation. If the tube is thin-walled tube, i.e. the mean radius, \( R \), is much greater than the wall thickness, \( h \), the effect of the longitudinal force, \( N \), and shear force, \( V \), can be negligible compared with that of the bending moment, \( M \). However, the \( R \) and \( h \) of the specimen used in this study, are 2.5 and 1 mm, respectively. Therefore, we have to take \( M, N \) and \( V \) into account. The strain energy absorbed in a quadrant, \( U_Q \), can be written by

\[
U_Q = \frac{1}{2} \left( \frac{M^2}{2EI} + \frac{N^2}{2LhE} + \frac{MN}{LhER} + \frac{V^2}{2LhG} \right) R d\phi
\]

where \( E \) and \( G \) (= \( E/(2(1+\nu)) \)) are the Young's modulus and the shear modulus of the ceramic tube and \( I \) is a value corresponding to the second moment of area for a straight beam, given by \( Lh^3/12 \). \( \alpha \) is a numerical factor by which the average shear stress must be multiplied to obtain the shear stress at the centroid of the cross section, e.g. \( \alpha = 3/2 \) for a rectangular cross section and \( \alpha = 4/3 \) for a circular cross section. Here, we adopted \( \alpha = 1.2 \) as used in the reference[18]. From Fig. 7(b), \( M, N, \) and \( V \) are given by

\[
M = M_0 - \frac{PR}{2} \sin \phi, \quad N = \frac{PR}{2} \left( \frac{1 - h^2}{12R^2} \right), \quad V = -\frac{P}{2} \cos \phi
\]

where \( M_0 \) is obtained by using equ. (2) and a boundary condition, which is \( \partial U_Q / \partial M_0 = 0 \) at the vertical cross section A-A' in Fig. 7(b). The total strain energy of a circular tube, \( U \), is equal to \( 4U_Q \) and the deflection, \( \delta \), can be given by \( \delta = (\partial U / \partial P) \). Therefore, \( \delta \) may be expressed by

\[
\delta = \frac{3\pi R^3}{3h^2 L} \left\{ 1 - \frac{h^2}{12R^2} + 2\alpha(1+\nu) \frac{h^2}{12R^2} - \frac{8}{\pi^2} \left( 1 - \frac{h^2}{12R^2} \right)^2 \right\} \cdot P
\]

where

\[
\partial U / \partial P = 4U_Q = 4 \sum U_Q (\partial U_Q / \partial P)
\]
From equ.(4), we can obtain the expression for Young’s modulus as follows:

\[ E = \frac{3\pi R^3}{L h^3 \delta} \left( 1 - H^* + 4\alpha(1 + v)H^* - \frac{8}{\pi^2}(1 - H^*)^2 \right), \quad H^* = \frac{h^2}{12 R^2} \]  

(5)

The bending stress on the inner surface of a curved beam is given by the following equation[19],

\[ \sigma_A = \frac{M_0(h/2 - e)}{Lh e(R - h/2)} \]  

(6)

where \( e \) is the distance from the centroid axis to the neutral axis of the cross section, approximately given by \( e = RH^* \) for a rectangular cross section. As mentioned before, fracture occurs at location A when the load reaches the maximum load, \( P_{\text{max}} \). Therefore, if we consider \( M_0 = (P_{\text{max}} R / \pi)(1 - H^*) \) from equ.(3) and put it into equ.(6), the fracture stress, \( \sigma_{f*} \), can be determined by

\[ \sigma_{f*} = \frac{P_{\text{max}}}{\pi L h} \frac{(1 - H^*)(h/2 - RH^*)}{H^*(R - h/2)} \]  

(7)

By using equ.(5) and equ.(7), Young’s modulus and fracture stress can be obtained from the results of lateral compression tests.

The analysis shown above is based on elementary static curved beam theory. In general, it seems to be difficult to apply the beam theory to the beam with relatively large curvature and thickness such as the mean radius \( R = 2.5 \) mm and the thickness, \( h = 1 \) mm. In the analysis, however, the slope of load-deflection curves up to the first peak and the bending stress at location A (see Fig.7(b)) are used to evaluate the Young’s modulus and fracture stress of ceramic tubes, respectively. We have already verified that both the slope of \( P-\delta \) curves and the bending stress at A obtained from equs.(4) and (7) agreed well with the results of FEM analysis[20]. Therefore, we think that there is no problem to use equs.(4) and (7). As mentioned in the section 3.1, \( E \) is not affected by loading rate. In the range of this paper, therefore, the equs.(4) and (7) may be used to the results of dynamic tests, as the first approximation.

3.3 Young’s modulus and fracture stress

Fig.8 shows Young’s modulus obtained from static and dynamic lateral compression tests. The open and closed circles indicate the static and dynamic data, respectively. In obtaining these data, the mean value of \( P/\delta \) in the region where the load increases linearly before the first peak (see Fig.4 and 6) was used. Since dynamic and static data scatter in roughly same area at each specimen length, Young’s modulus of Mullite tube is not affected by loading rate.

Young’s modulus observed in static tensile tests is also shown by a solid line for comparison. It was about 114 GPa. From the static data, the average values of Young’s modulus for 4, 7 and 10 mm specimens were about 102, 99 and 90 GPa, respectively. Thus, Young’s modulus obtained from lateral compression tests are a little lower than that in tension tests. The reason for this difference is not clear, however, one of the reasons considered is the accuracy in the measurement of deflection in lateral compression tests. In this study, the outer diameter of the specimen tube is 6 mm which was adopted to make tensile tests easy. Therefore, the deflection of the specimen in lateral compression became extremely small. An effort to raise the accuracy was made, but it might be not enough. Another point is that the average for 10 mm specimen is about 10% lower than the other two averages. As mentioned above, equ(5) is based on curved beam theory, in which uniform deformation along the tube axis is a fundamental assumption. The specimens originally included some inaccuracy in their shape and size. The effect of the inaccuracy on the deformation of a tube may be greater in a long specimen than in a short one. Thus, it appears to be more difficult for a longer specimen to achieve uniform deformation along the tube axis properly. This may be one of the reasons why the average of the...
shortest specimen is highest. From these discussion, the specimen with $L/2R \leq 1$ is recommended to estimate Young's modulus of ceramic tube from lateral compression tests.

Fracture stresses of tubular specimens in static and dynamic lateral compression were calculated by eqs.(7) and shown in Fig.9. In the figure, open circles, closed circles, a dotted line and a broken line are, respectively, static data, dynamic data and average lines of dynamic and static data. The solid line indicates the catalogue value of the bending strength of Mullite tube. As shown in static data, the average fracture stresses of specimen with lengths of 4 and 10 mm are about 157 MPa and 133 MPa, respectively, i.e. the average fracture stress of 10 mm specimen is about 15 % lower than that of 4 mm specimen. The number of defects involved in a longer specimen is greater than that in a shorter specimen. Therefore, the probability of involving larger defects in longer specimens becomes higher. This may be one of the reasons of the difference between average fracture stresses of 4 and 10 mm specimens. Similar tendency is also seen in dynamic data. In each specimen length, however, the average fracture stress at dynamic rate is about 1.6 times greater than that observed in static tests. This may result from the effect of loading rate.

4. CONCLUSIONS

In order to investigate the deformation of Mullite ceramics tube in lateral compression, a series of experiments was carried out at dynamic and static rates. Static tensile tests were also performed to determine the basic mechanical properties of Mullite ceramics tube. Principal results obtained are:

1. Expressions for Young's modulus and fracture stress can be derived theoretically by considering the deformation of thick tube in lateral compression. By using the expressions, Young's modulus and fracture stress of ceramics tube are obtained from the results of lateral compression tests.

2. The effects of loading rate on Young's modulus were not found. In estimation of Young's modulus from lateral compression tests, the specimen with $L/2R \leq 1$ is recommended, although it may be slightly underestimated comparing with tension tests.

3. The average fracture stress obtained from dynamic lateral compression tests was about 1.6 times greater than that obtained from static tests. Therefore, the strength of Mullite tube is affected by loading rate. It was also found in static data that the fracture stress agreed well with the bending strength of the Mullite tube.

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